



May 15, 2007

**Pressure Independent Modulating Control
Valves:
Impact of Design or Greater ΔT
on Chilled Water & Hot Water Systems**

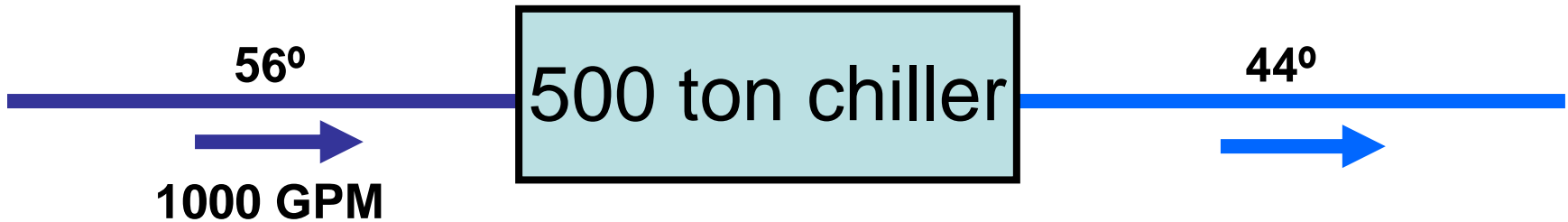
Presented by:
David Pleasants

$$BTUH = 500 \times GPM \times \Delta t$$

$$Tons = \frac{500 \times GPM \times \Delta t}{12,000}$$

$$\frac{500 \times 1000 \times 12}{12,000} = 500 Tons$$

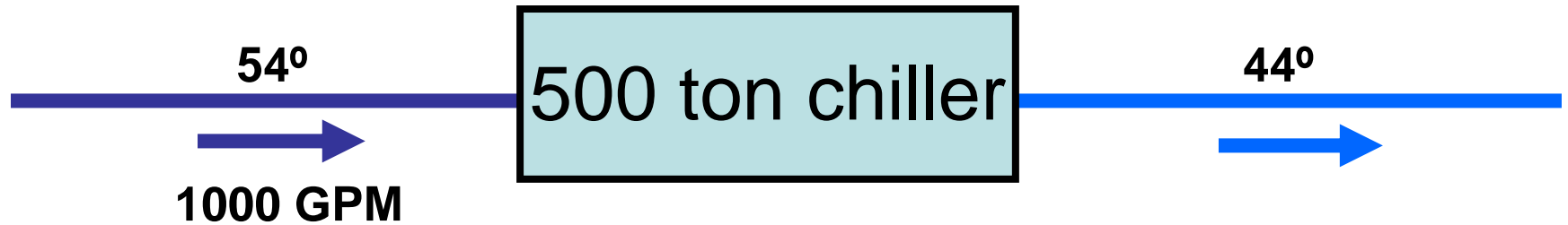
Design Conditions:



$$\frac{500 \times 1000 \times 12}{12,000} = 500 \text{ Tons}$$

Chiller is fully loaded

Low ΔT :

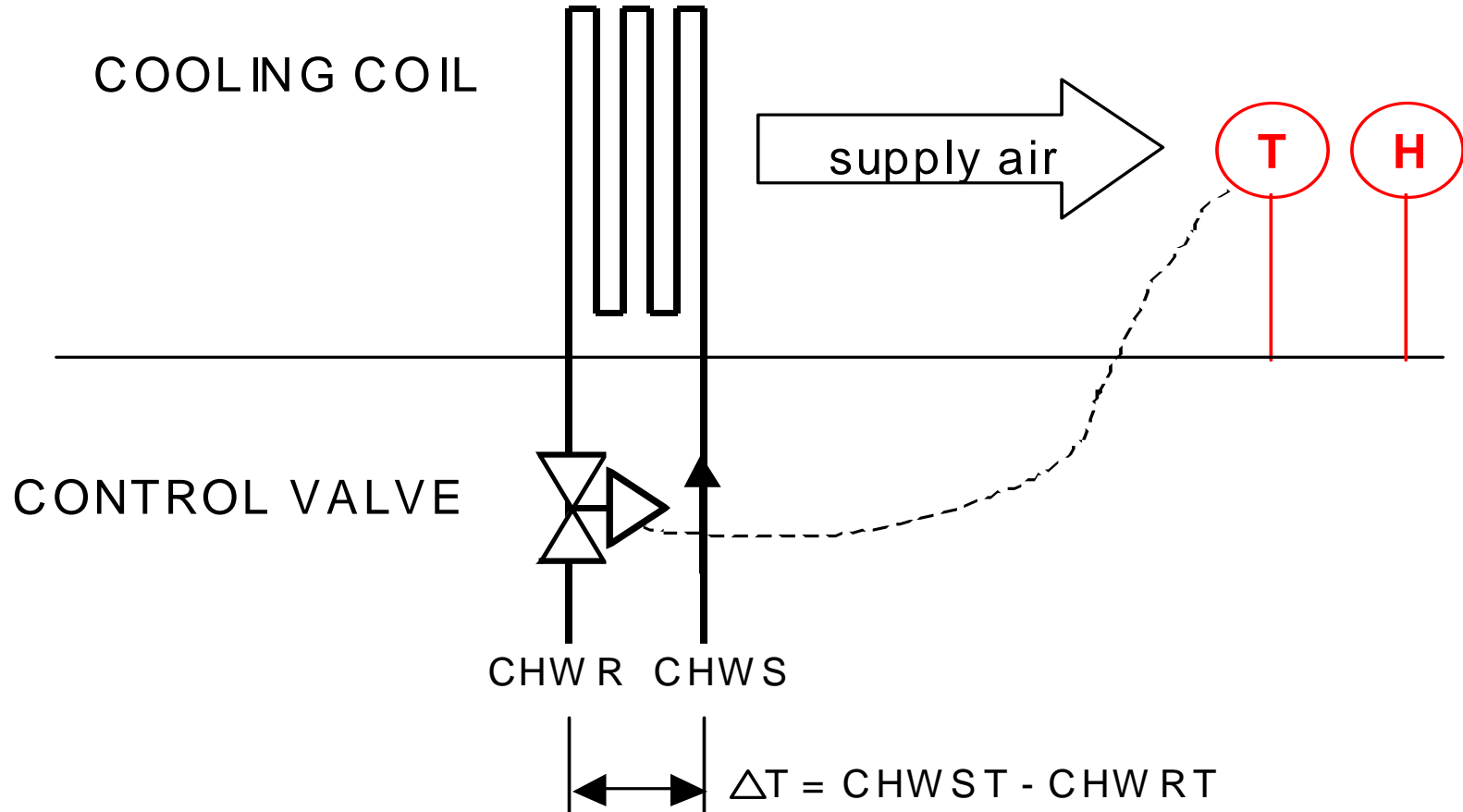


$$\frac{500 \times 1000 \times 10}{12,000} = 417 \text{ Tons}$$

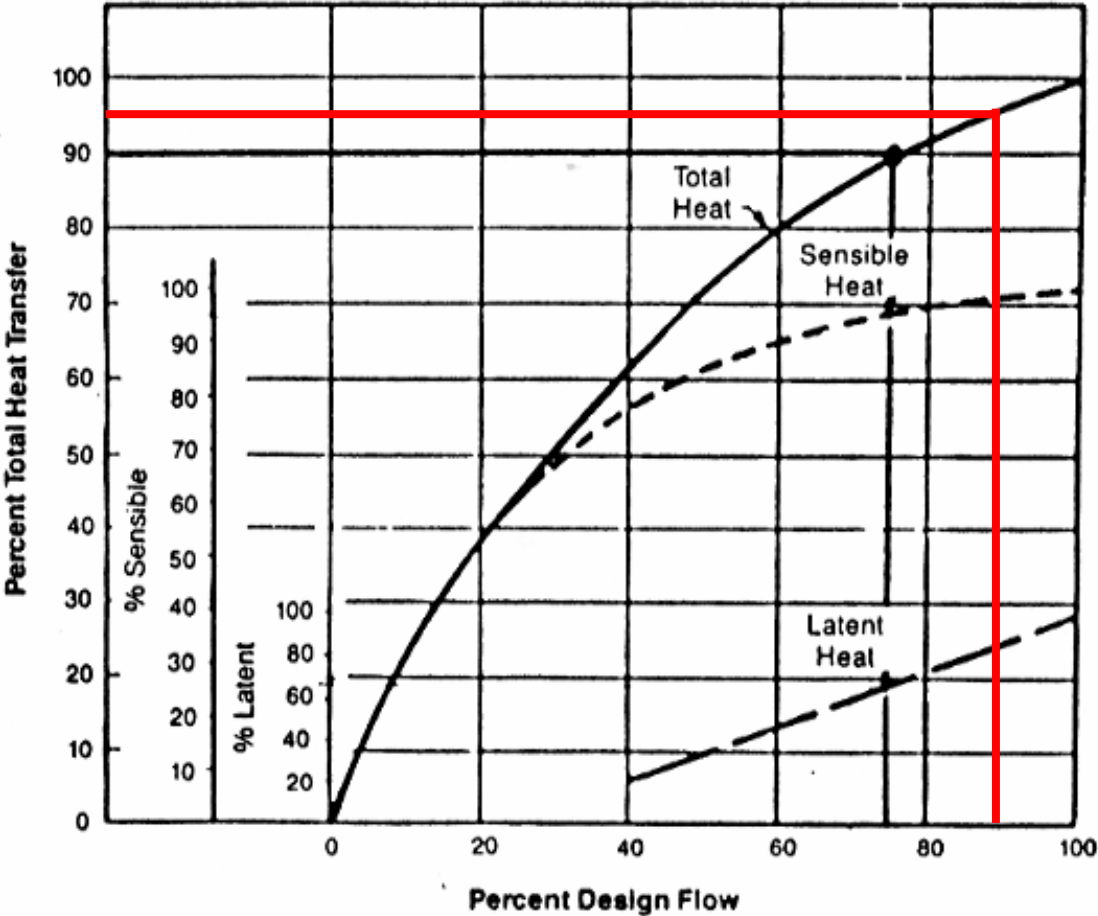
Chiller is 83% loaded

A.H.U. / Coil Performance:

What sets demand for chilled water?

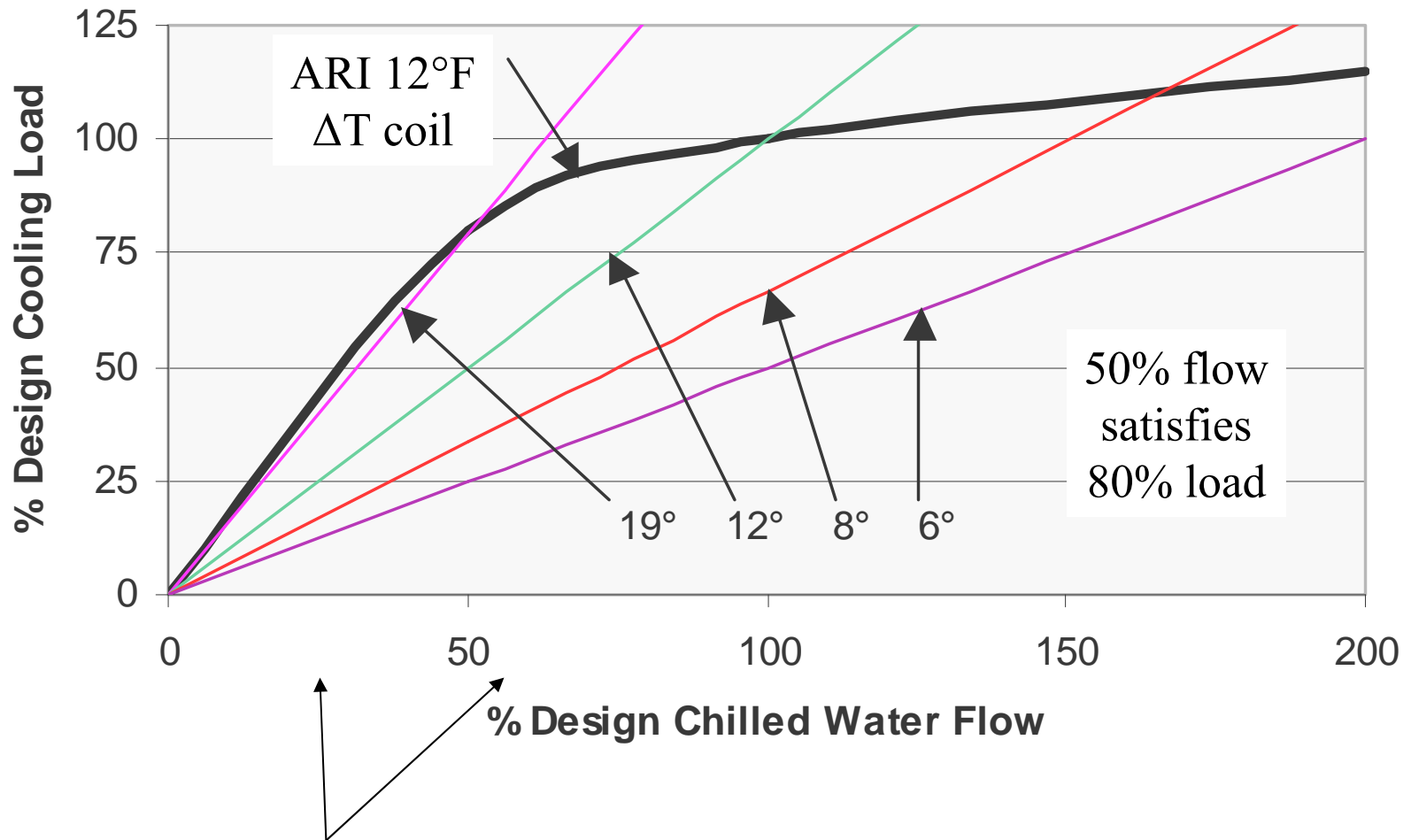


Cooling Coil Heat Transfer vs. Water Flow:



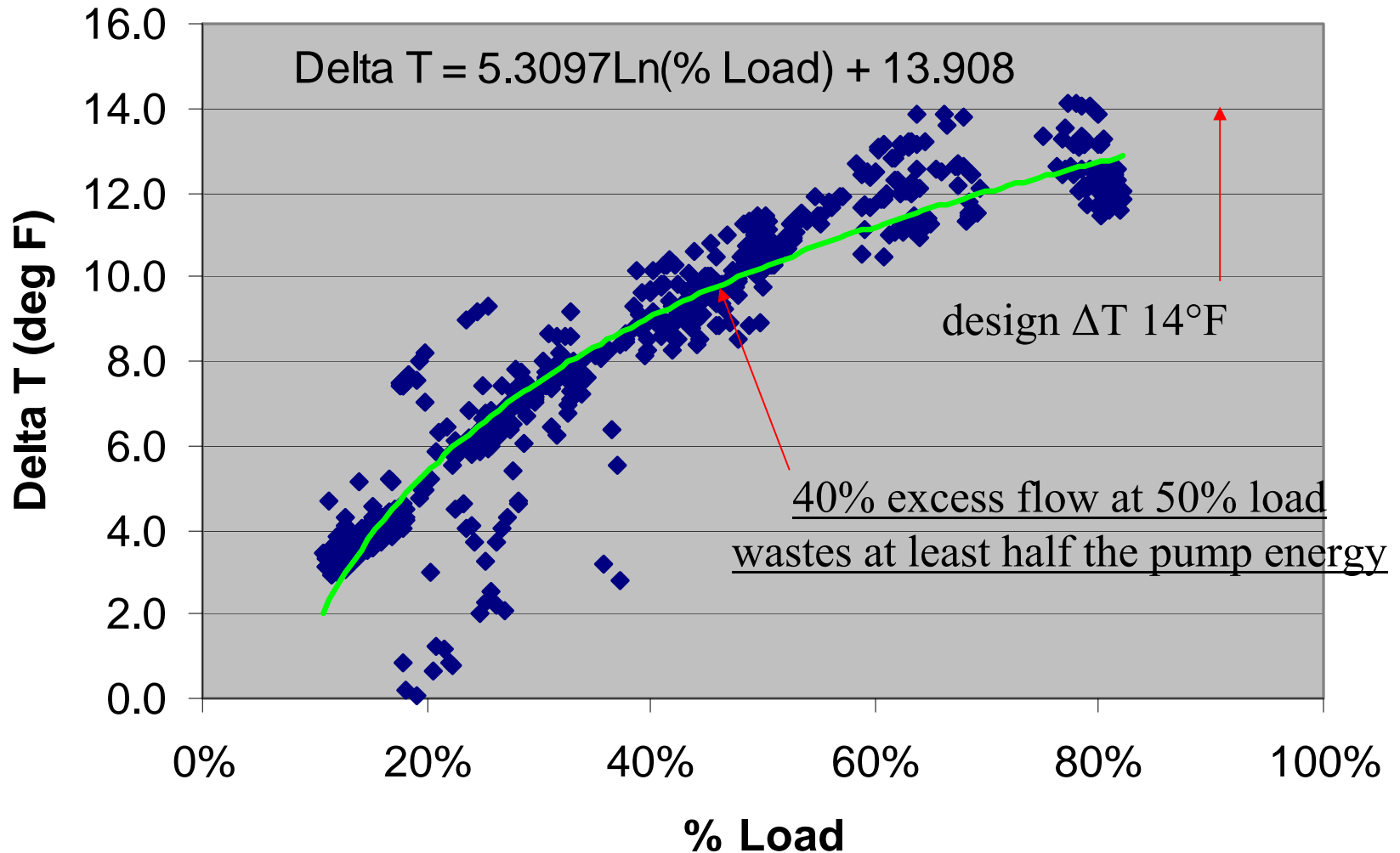
2003 ASHRAE Applications Handbook, p 37.8

How should a cooling coil perform?

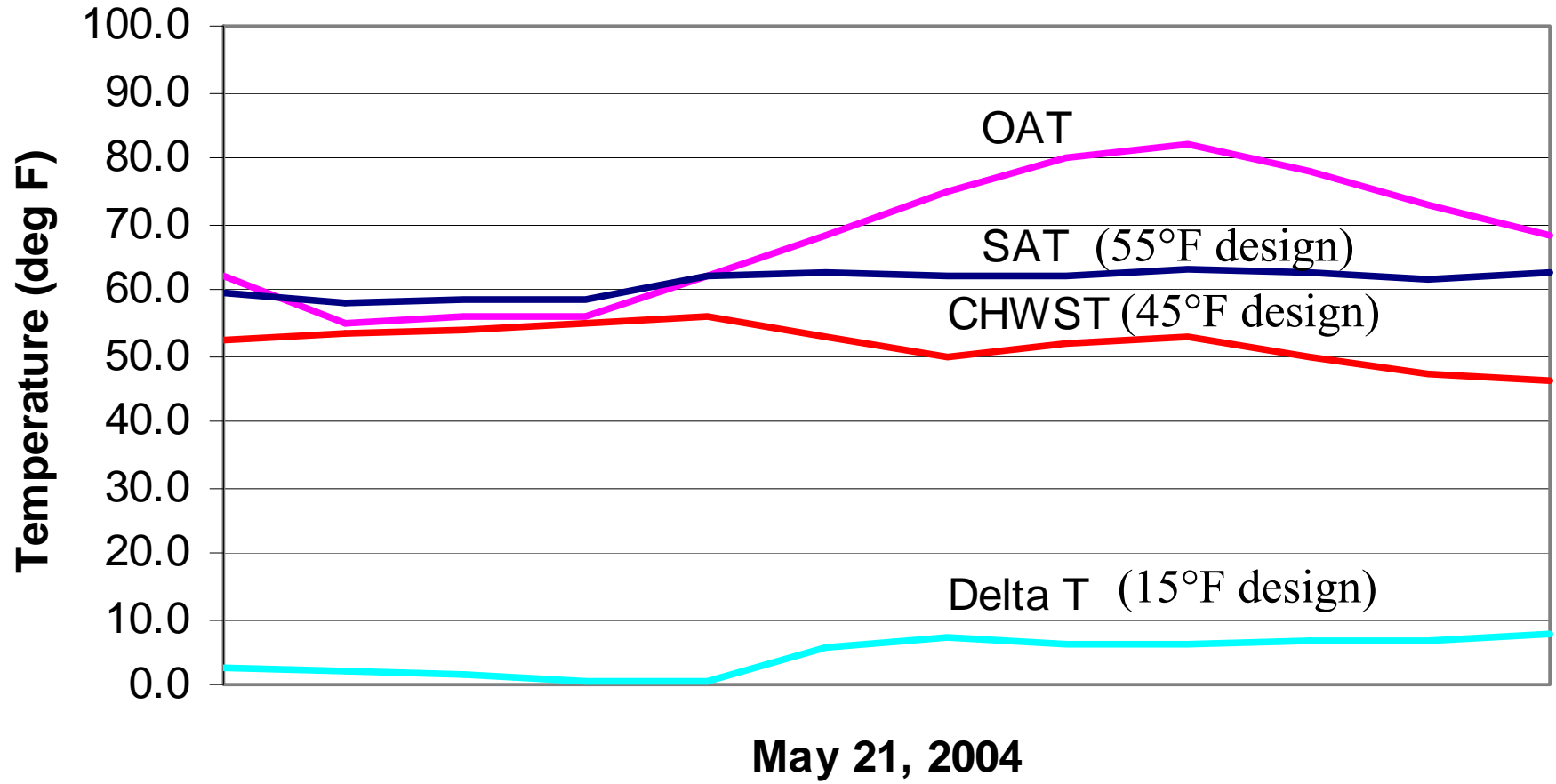


Control valves must modulate well in this range
for good system performance!

Typical system with low ΔT :



Trend of poor performance



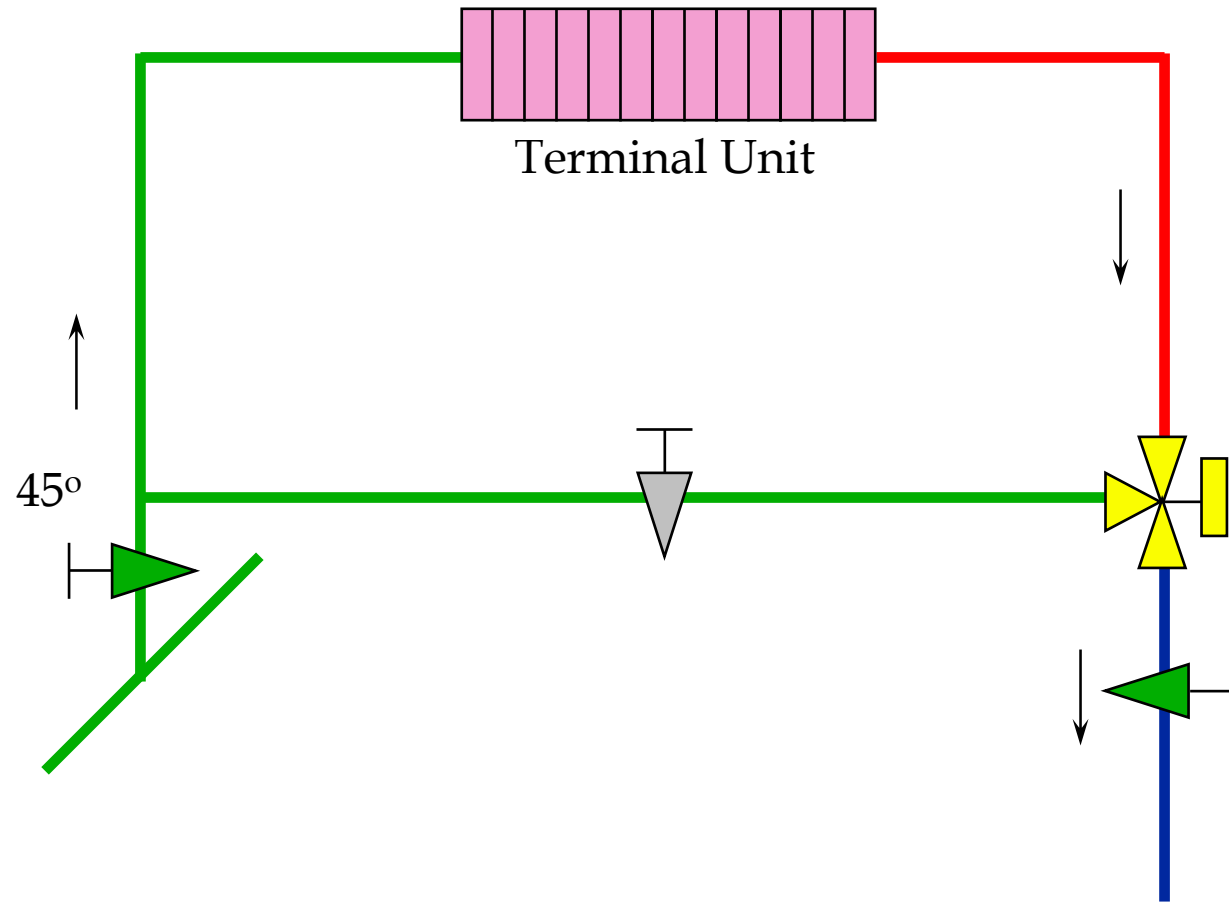
Why should ΔT exceed design?

- ΔT is a parameter used in design at peak load to size chillers, pipes, pumps, and cooling coils.
- In operation at all loads, ΔT is the most important measure of total distribution system performance.

$$\frac{\text{ton}}{\text{gpm}} = \frac{\Delta T}{24} \quad \longleftarrow \begin{array}{l} \text{Maximize available capacity} \\ \text{Increase tons/gpm} \\ \text{(chillers, pumps, pipes, coils)} \end{array}$$

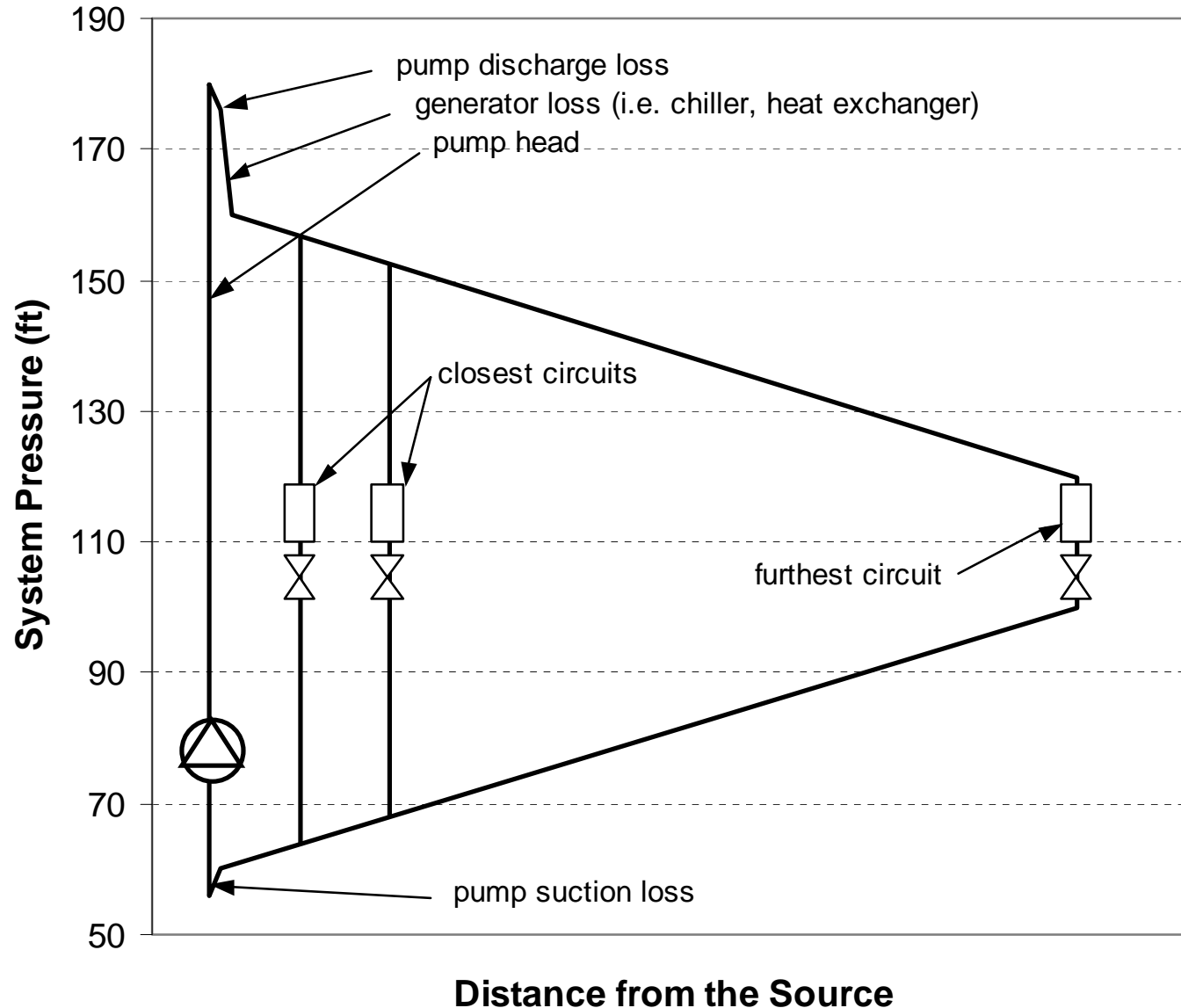
$$\frac{\text{gpm}}{\text{ton}} = \frac{24}{\Delta T} \quad \longleftarrow \begin{array}{l} \text{Minimize energy use} \\ \text{Decrease gpm/ton} \\ \text{(chiller plant, distribution, fans)} \end{array}$$

What Will Cause ΔT
To Be Less Than Design
AT THE COOLING COIL?



Three-Way Valve

What do pumps and control valves do?



Pumps generate system pressure.

Control valves must consume pressure to modulate flow.

WHAT CAN CAUSE LOW ΔT ?

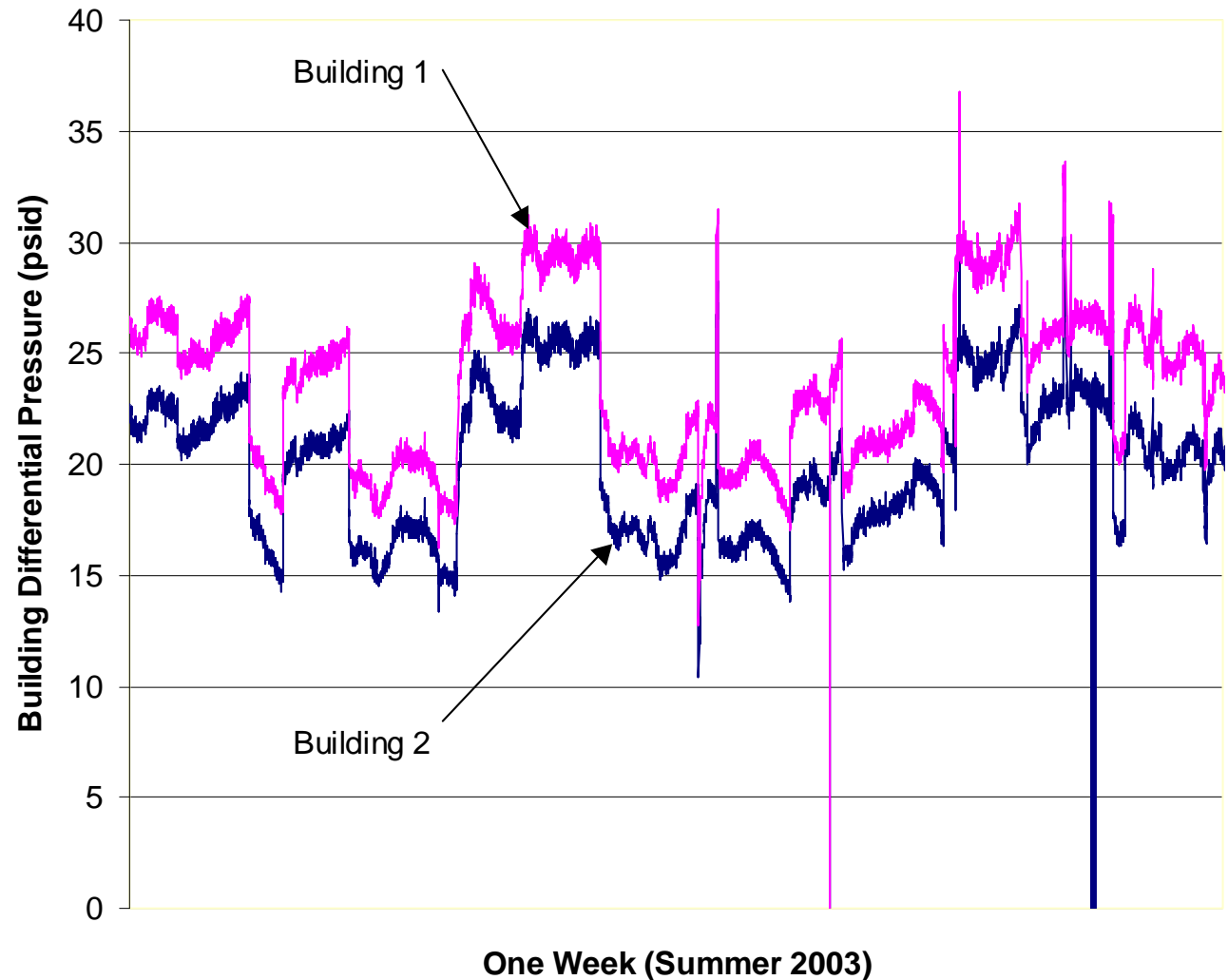
- Dirty or broken coils
- Not enough heat transfer surface
- Laminar flow (not really)
- Real time pressure fluctuations
- Rule of thumb control valve sizing
- Control valve rangeability and system turndown
- Rising supply water temperature to coils

real time system pressure fluctuations

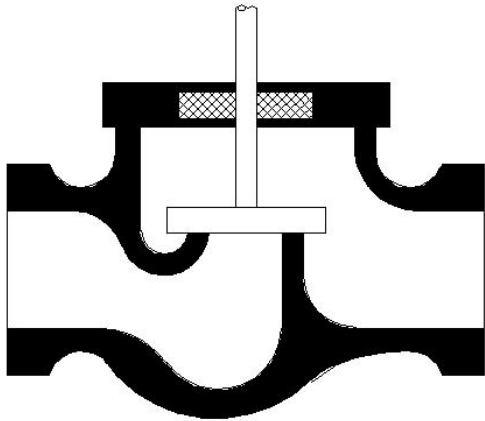
Cooling loads
rise and fall

Control valves
open and close
to meet demand

Pumps start,
stop and
change speed



Conventional control valves must react to both load and differential pressure changes.



Conventional Control Valve



Flow varies as differential pressure varies.

It takes TIME for a conventional control valve & actuator to react to differential pressure changes, with or without a load change.

Conventional control valve sizing issues:

- Rule of thumb valve sizing
- Hydraulic profile (today)
- Hydraulic profile (tomorrow)

Rule of thumb control valve sizing:

- Line size or one size smaller.
- 5 psi drop (anywhere in system).
- By maximum pump head for the loop.
- Same pressure drop as coil served.
- For authority (with limited DP data).

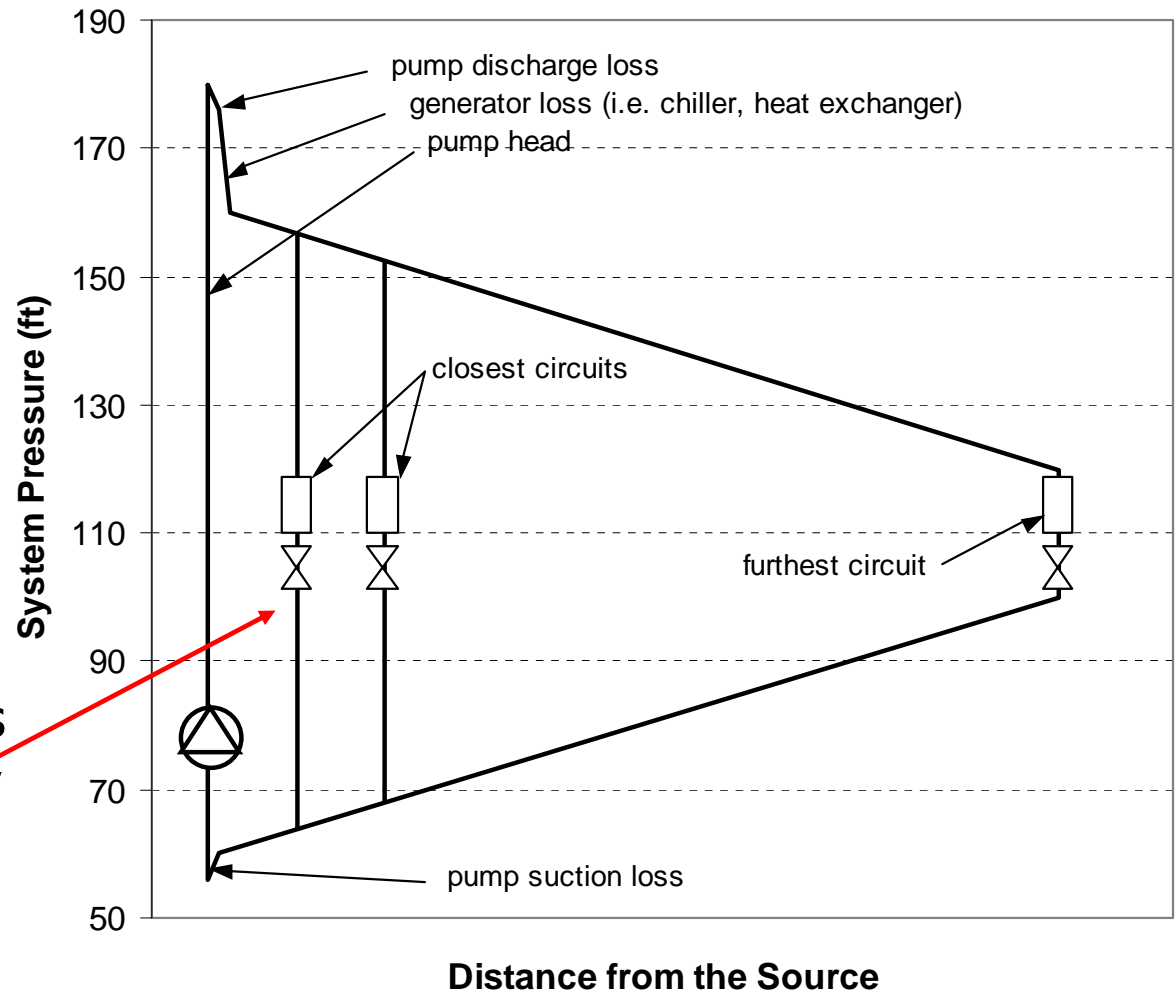


The problem commonly starts with uncertainty
in the hydraulic profile of the system

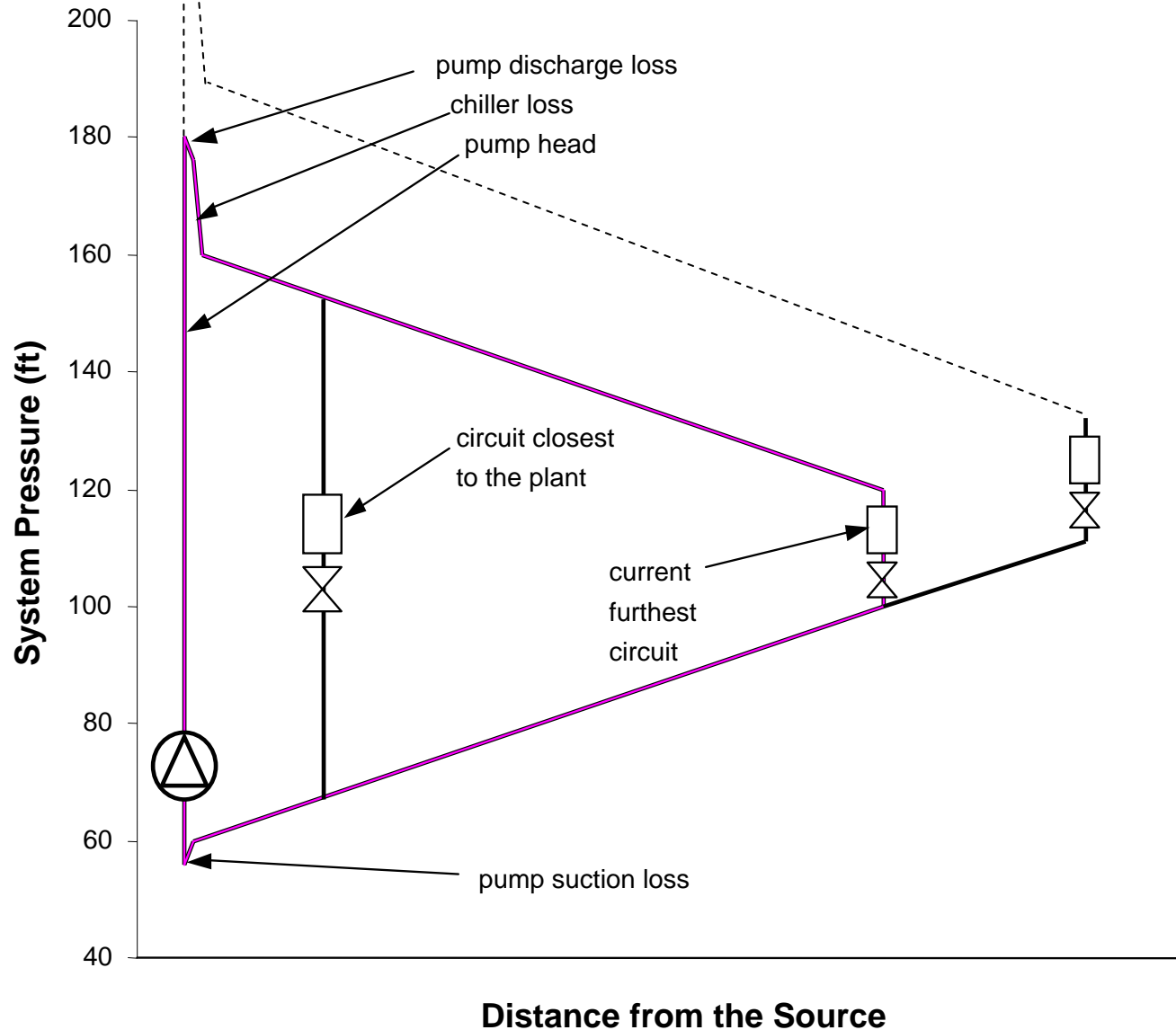
What happens when typical control valves are sized by minimum pressure drop?

Valve Rangeability vs. System Turndown

Conventional control valves close to the plant may only use a small portion of their full range.



What happens (tomorrow) to the hydraulic profile when loads are added or changed?

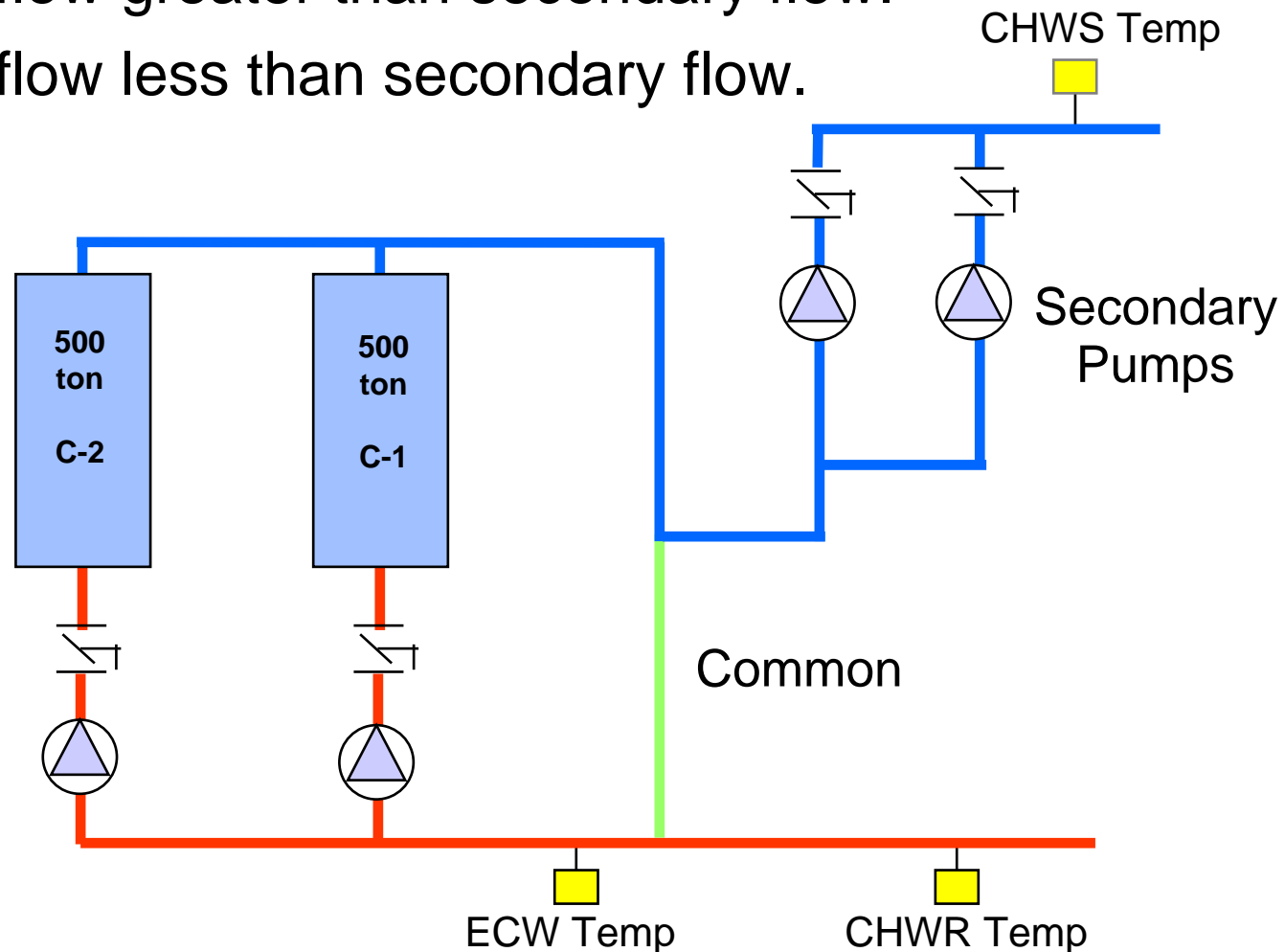


3 Possible Conditions of conventional primary / secondary production plant:

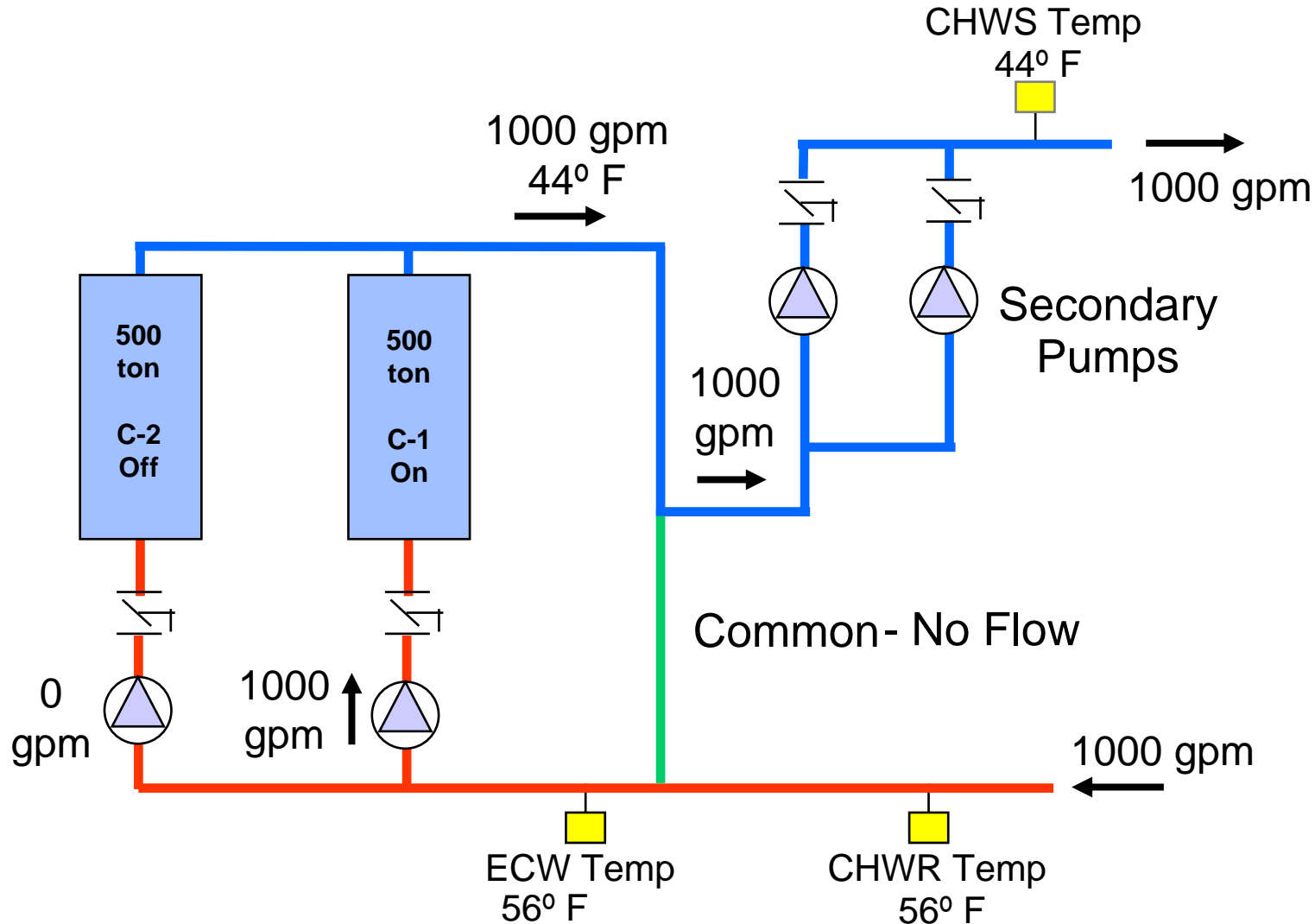
Primary flow equal to secondary flow.

Primary flow greater than secondary flow.

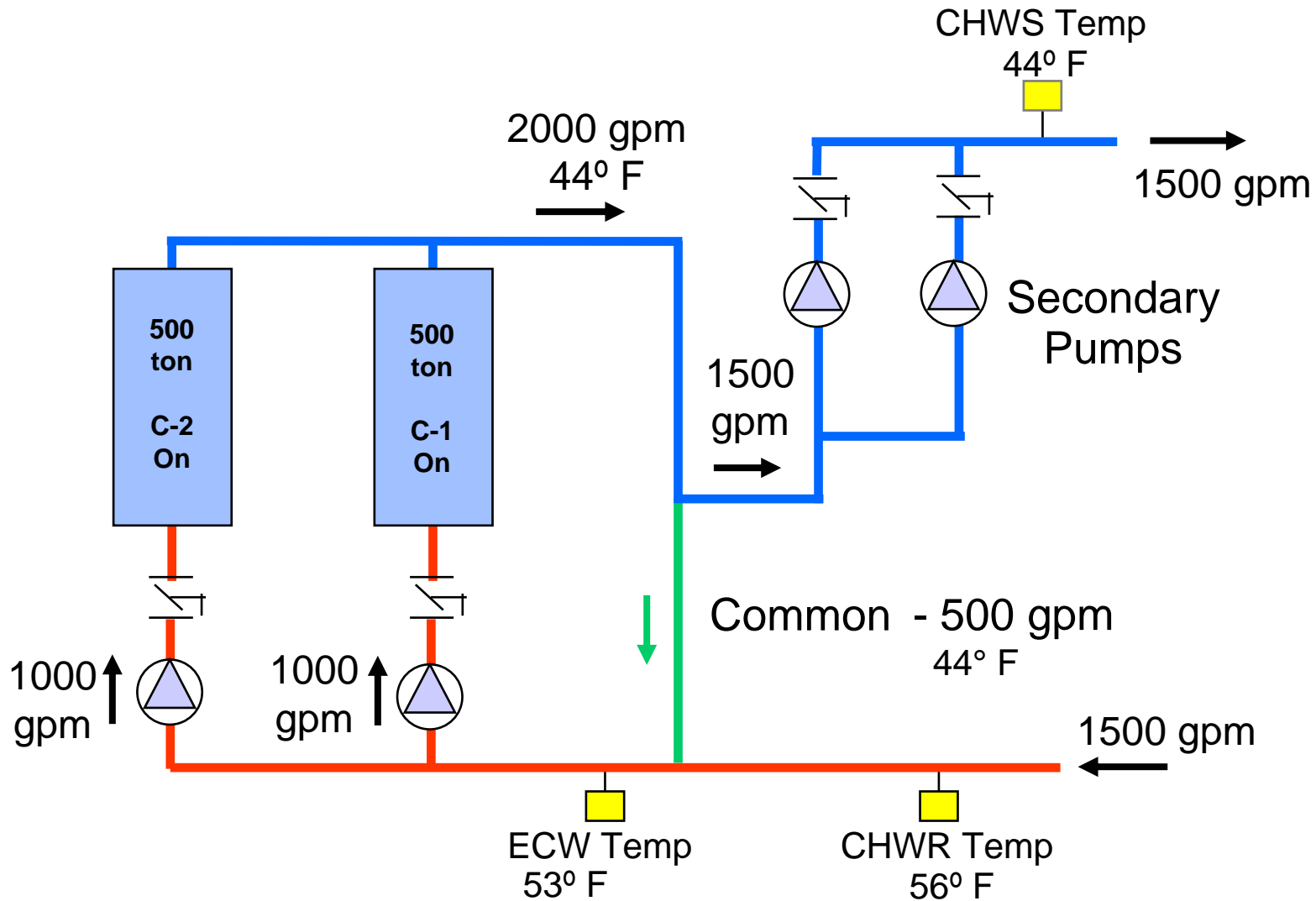
Primary flow less than secondary flow.



Primary flow equal to secondary flow:

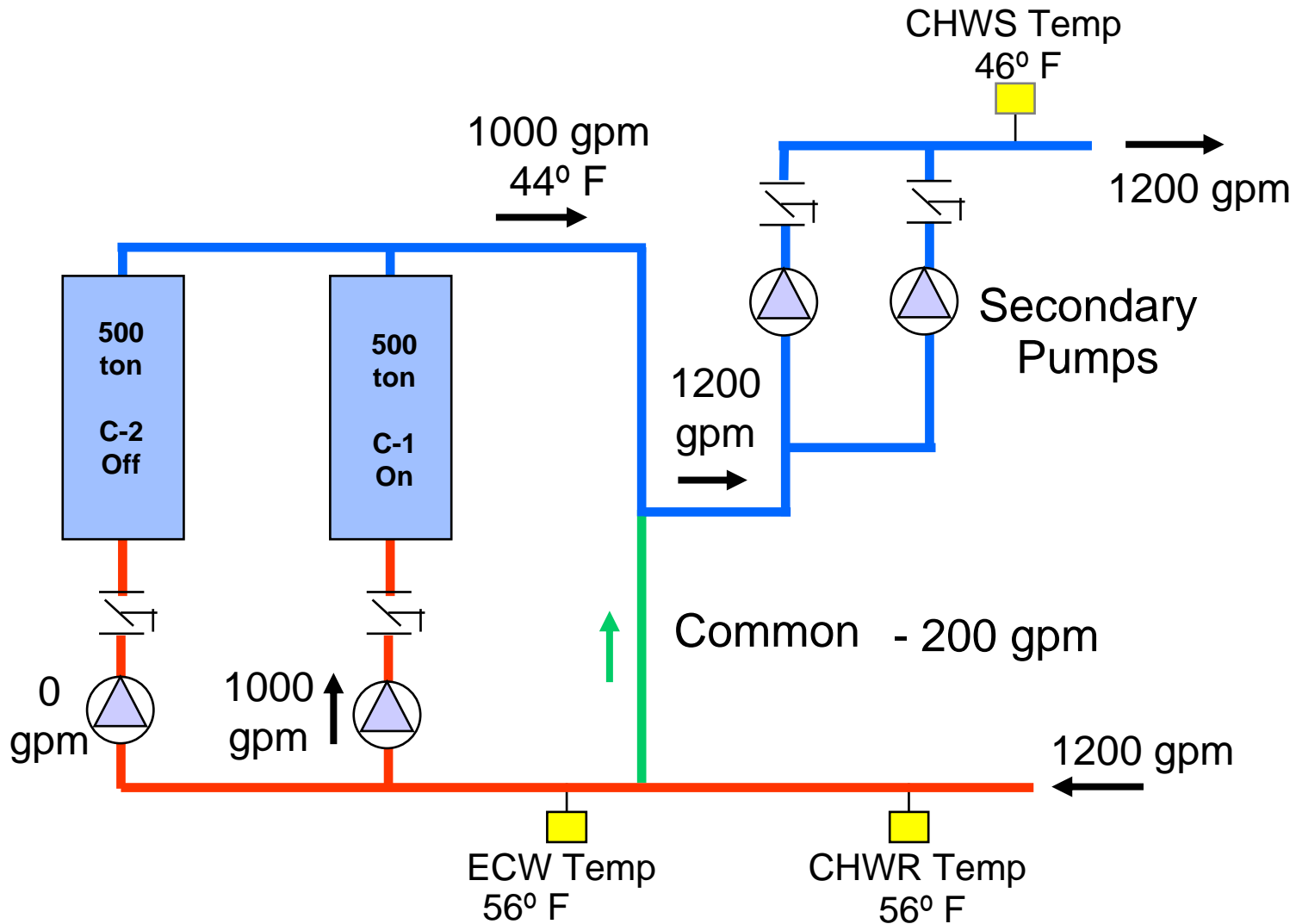


Primary flow greater than secondary flow:



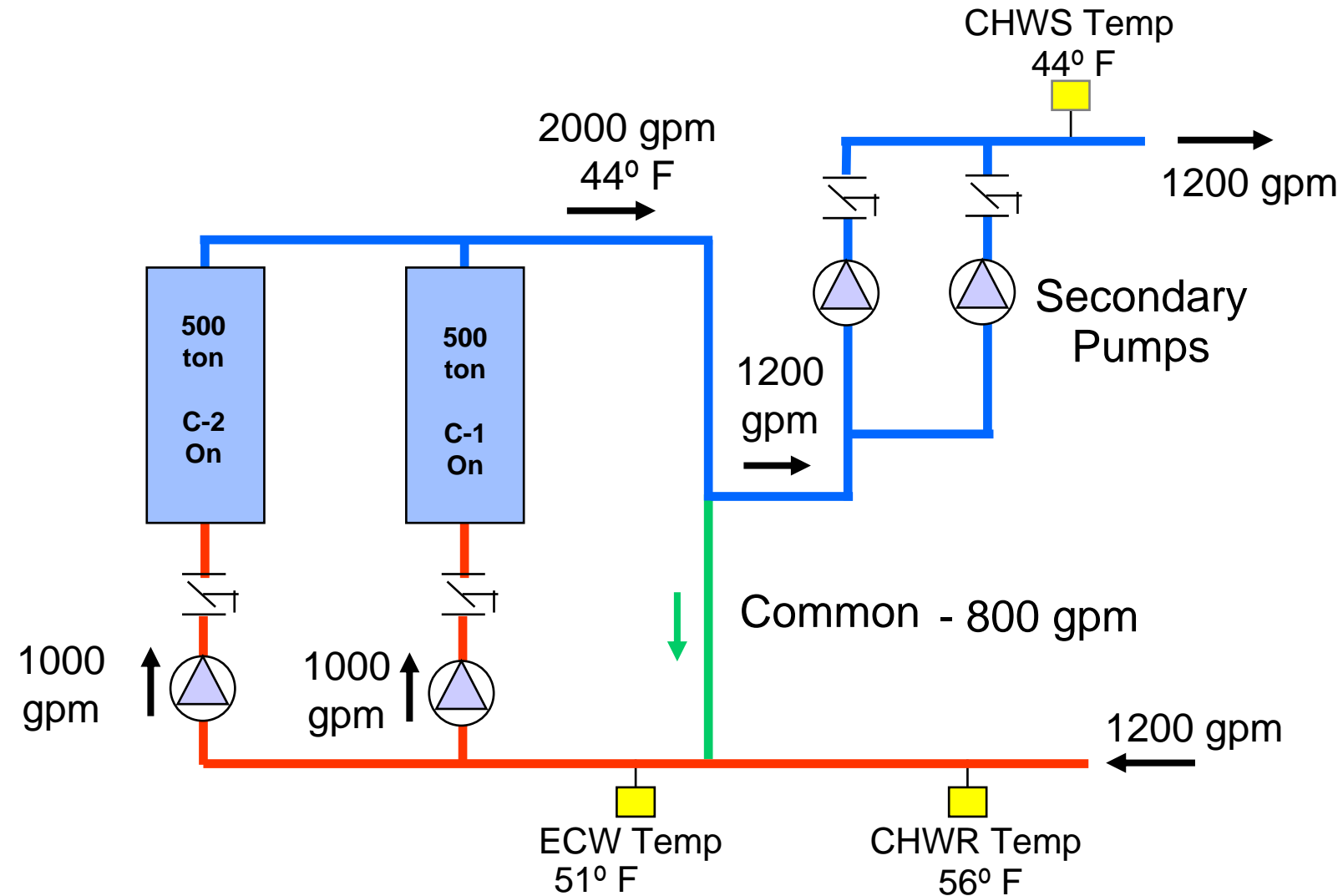
Each chiller is 75% loaded (9°/12°)

Primary flow less than secondary flow:

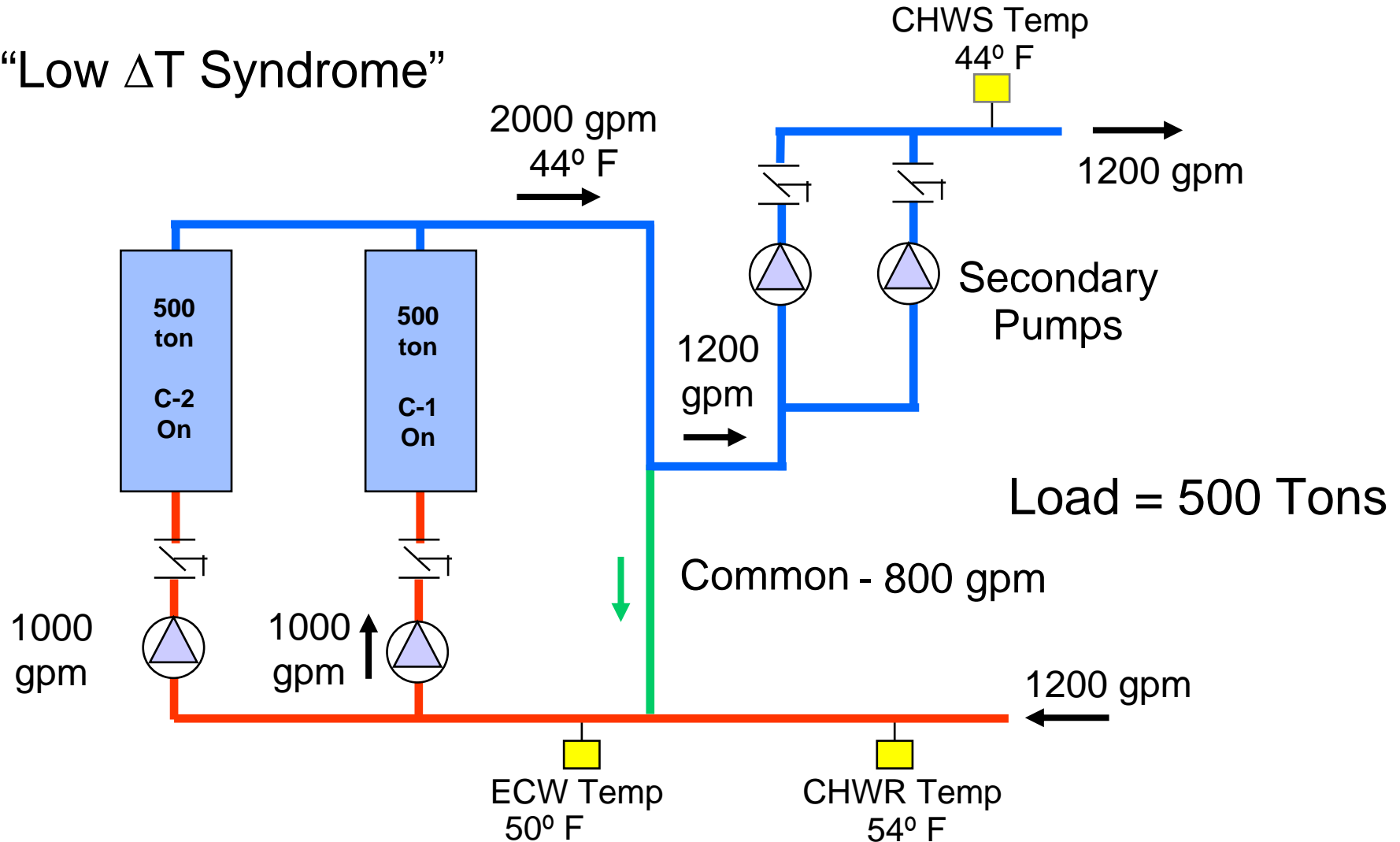


Stage on Chiller #2, to get....

Note: when we turned on primary pump #2, the primary flow increased to 2000 gpm, but secondary flow remained at 1200 gpm.



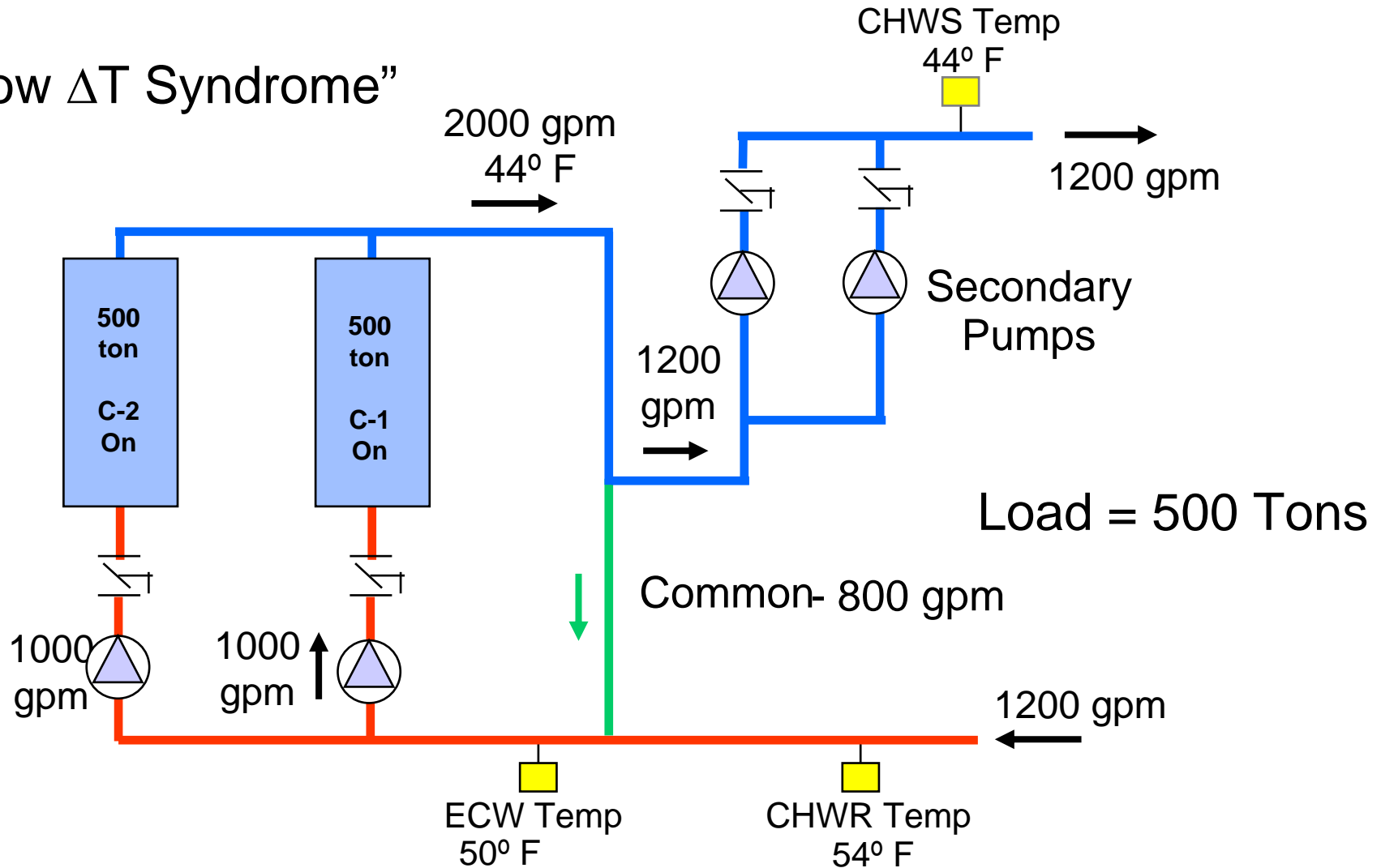
“Low ΔT Syndrome”



With 54° F return water from the system, we cannot meet the 500 ton load with one chiller

Let's look at wasted energy in this example because of the low Δt ²⁹...

“Low ΔT Syndrome”



- Our pump bhp on the primary side is over twice than what it would be with a 12° Δt
- We are pumping 20% more flow on secondary side. Secondary pump bhp increase of $\approx 60\%$
- Running 2 chillers instead of 1; 2 towers instead of 1; and 2 condenser pumps instead of 1
- Our chiller plant cannot meet a 1000 load with the low Δt ...

Changes we can make to meet the 1000 ton load in our example

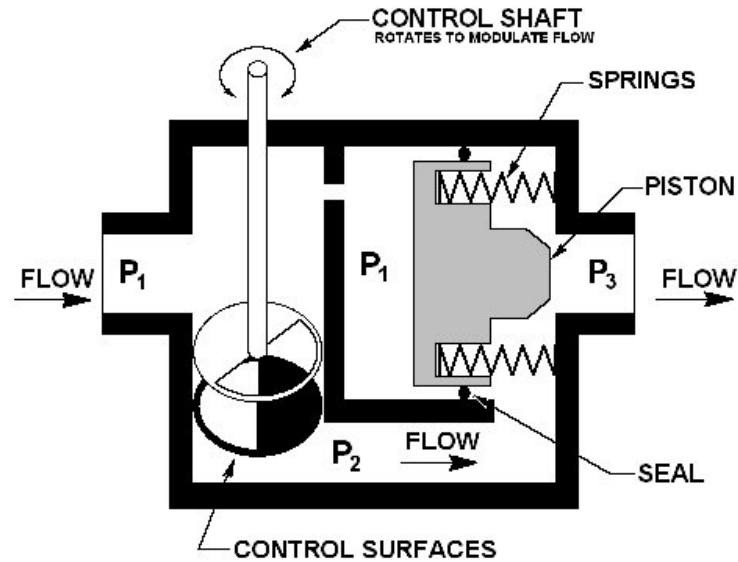
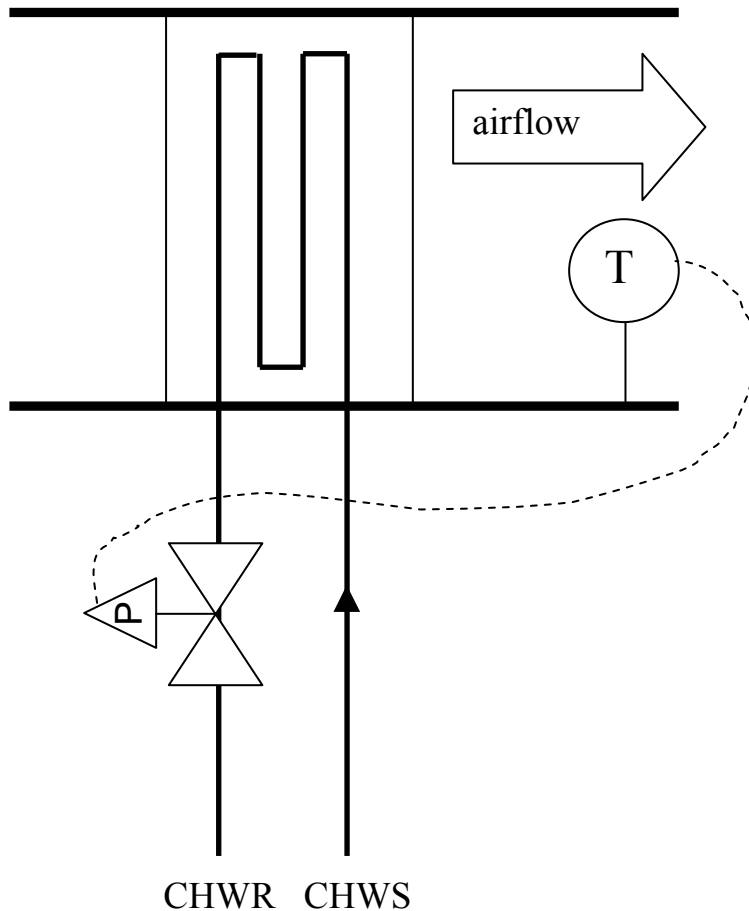
- Fix the low Δt problem out in the system (low Δt is caused by a problem out in the system... it is *not* caused by primary-secondary)
- Buy a 170-ton chiller

PREVENTING LOW ΔT :

High Quality Pressure Independent
Modulating Control Valves Are a
Solution to Low ΔT

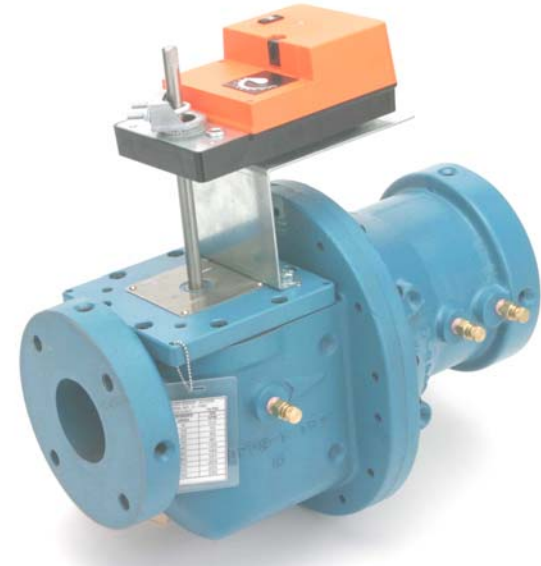
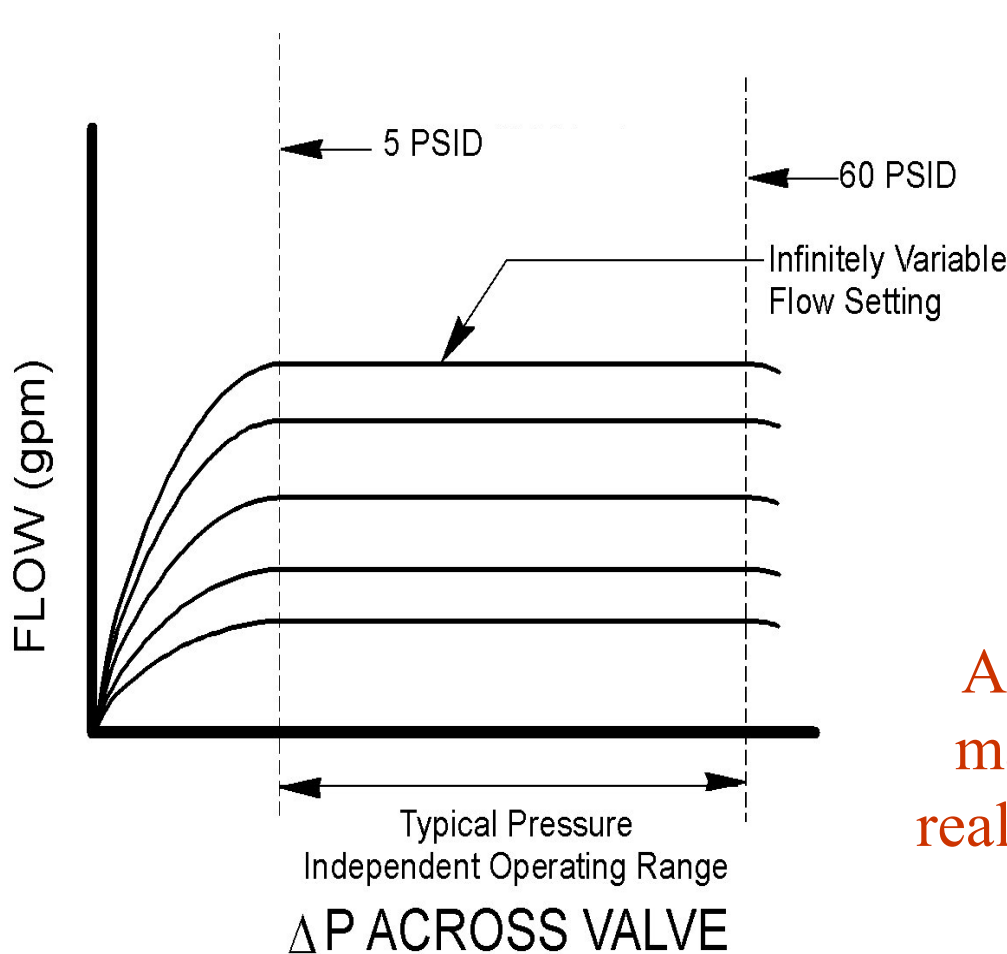
Pressure Independent Modulating Control

Valves only react to changes in the load



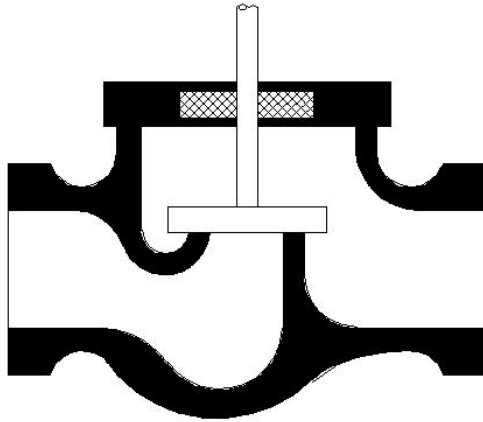
Flow is steadied by maintaining constant differential pressure across the control surface

Flow remains steady over a wide operating pressure range:



Applied at coils, these valves make the system **ROBUST** to real time fluctuations in pressure **AND** uncertainty in the hydraulic profile.

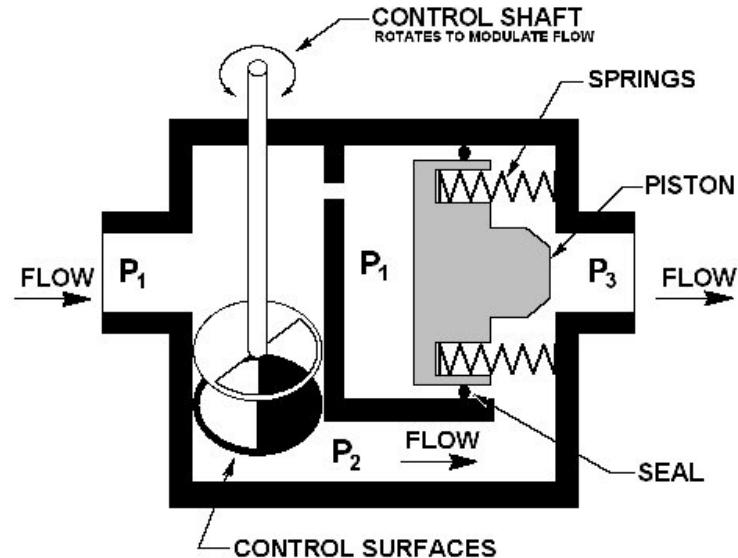
Pressure independent control valves don't care about pressures:



Conventional Control Valve

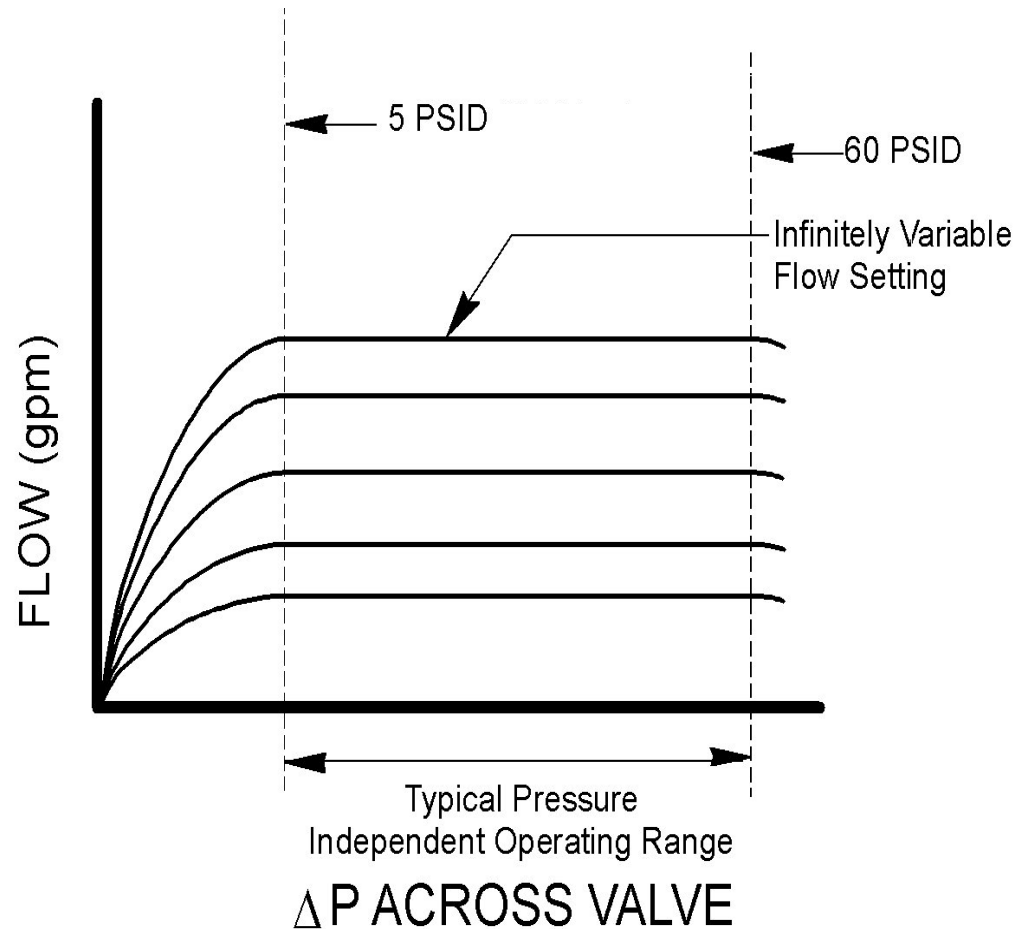


Flow varies as differential
pressure varies

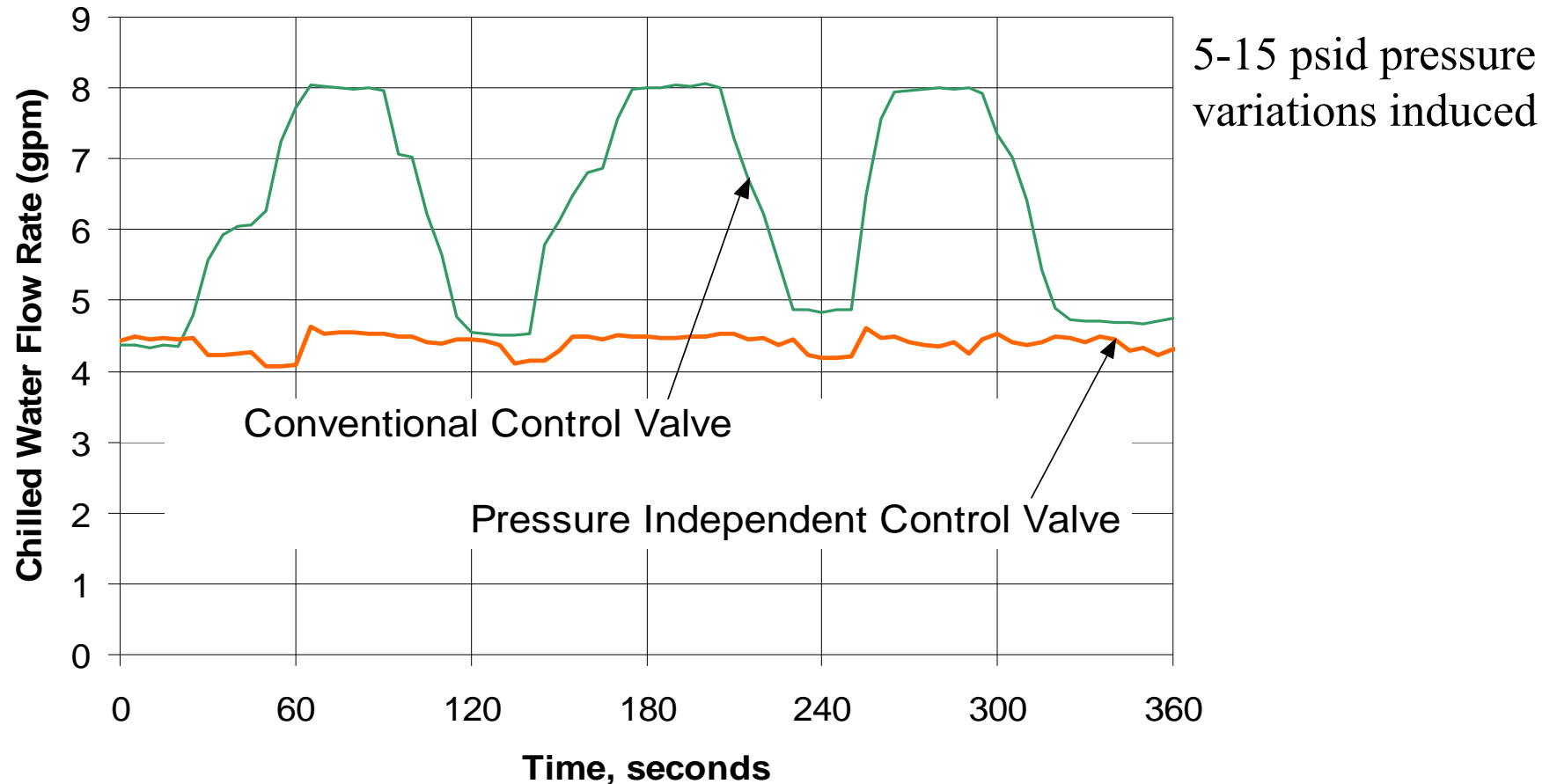


Flow is steadied by maintaining
constant differential pressure
across the control surface

Flow remains steady in a wide operating pressure range

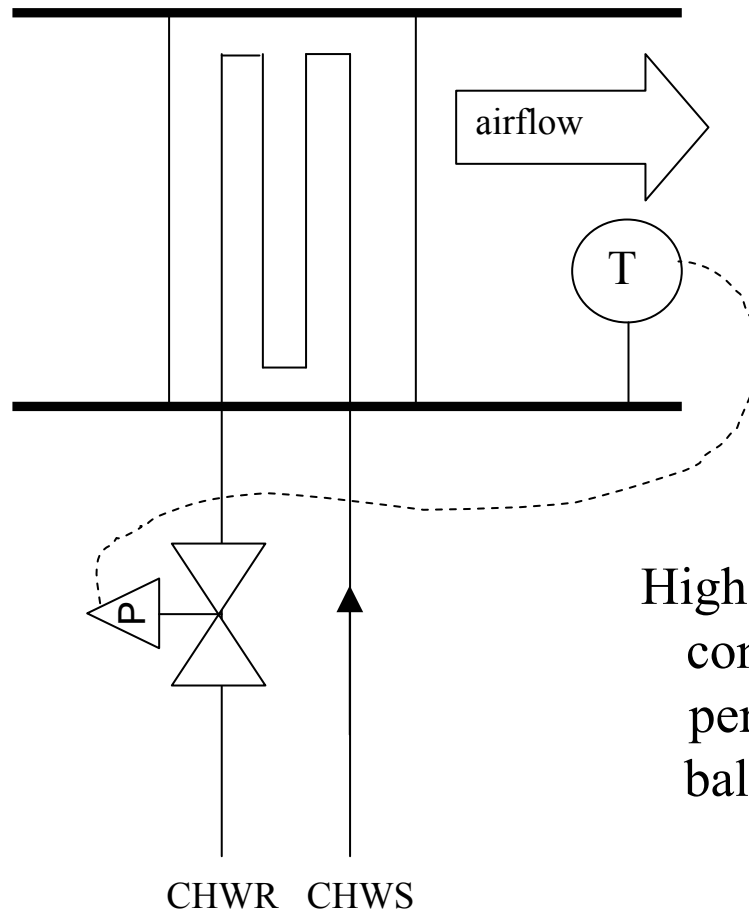


Pressure variations have no effect on the flow rate through cooling coils



IT TAKES TIME FOR A CONVENTIONAL CONTROL VALVE TO RETURN TO THE RIGHT POSITION TO SERVE THE COOLING LOAD. CONTROL VALVE “HUNTING” CONTRIBUTES TO LOW DELTA T

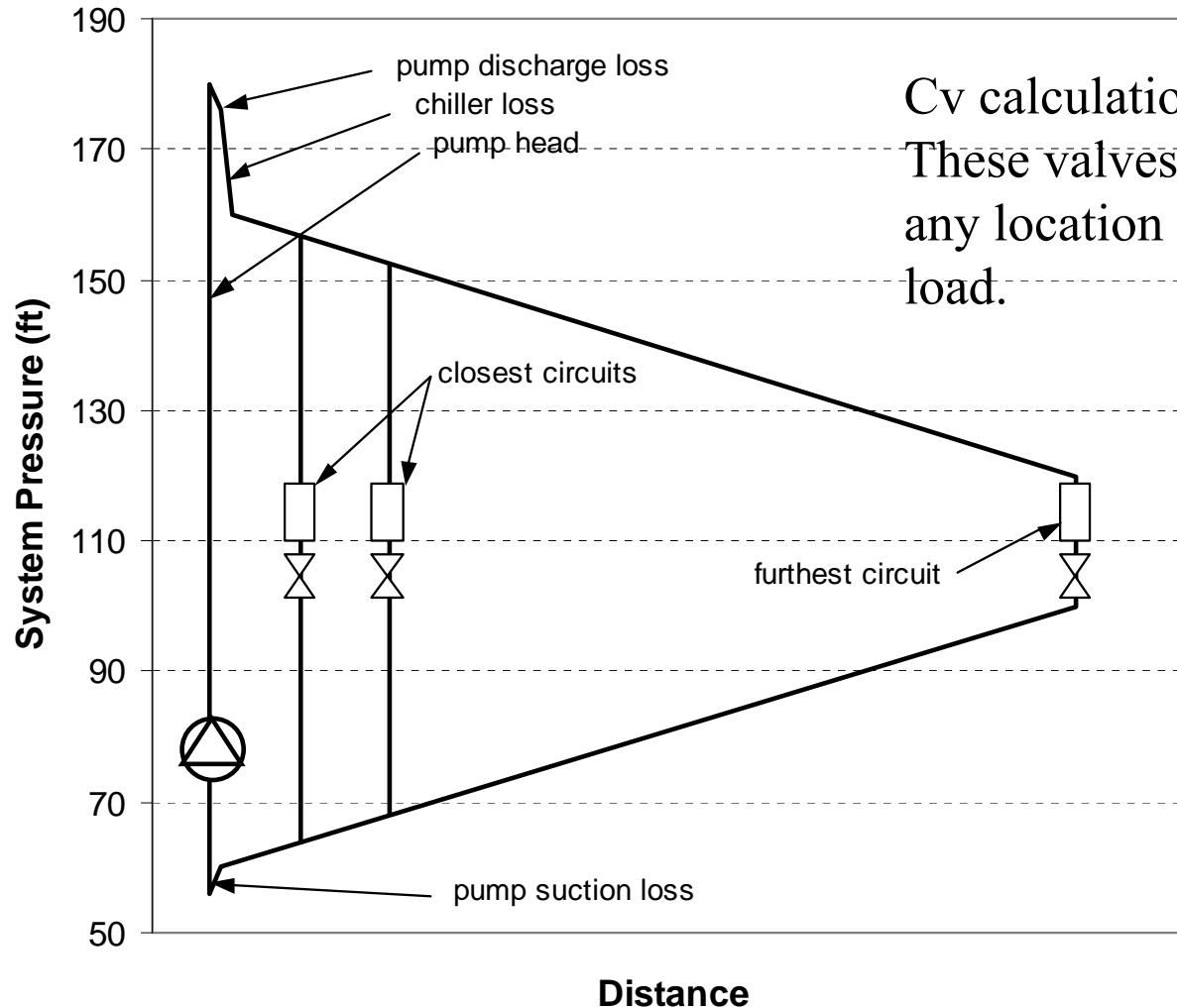
They are typically applied at terminal unit cooling coils:



Leaving air temperature thermostat alone usually sets the demand for chilled water flow.

High quality pressure independent control valve optimize ΔT performance and dynamically balances the system at all load conditions.

They are selected by flow rate,
not Cv and differential pressure -



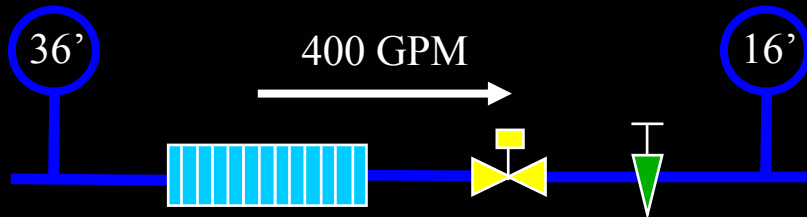
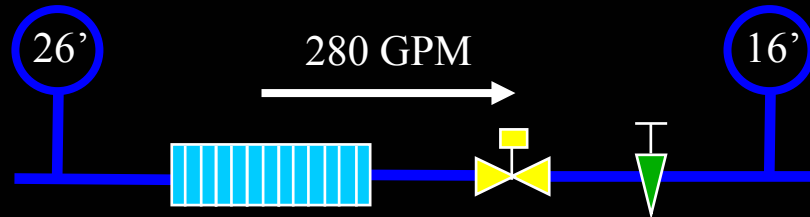
Cv calculations are not required.
These valves operate the same at
any location in the system at any
load.

System is robust
to future changes
in the piping, load
or hydraulic
profile.

Typical Part Load Day

Design Load = 500 GPM

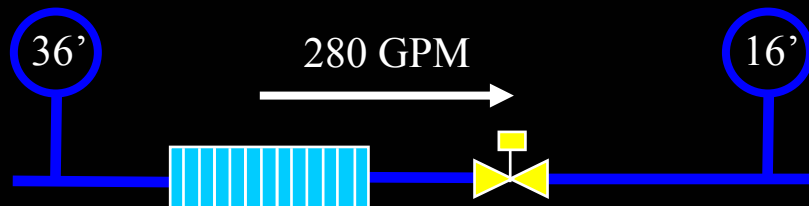
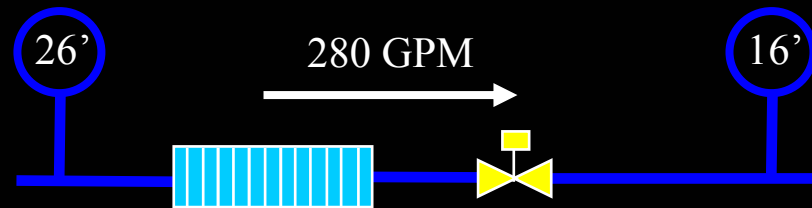
Conventional Control Valve



Typical Part Load Day

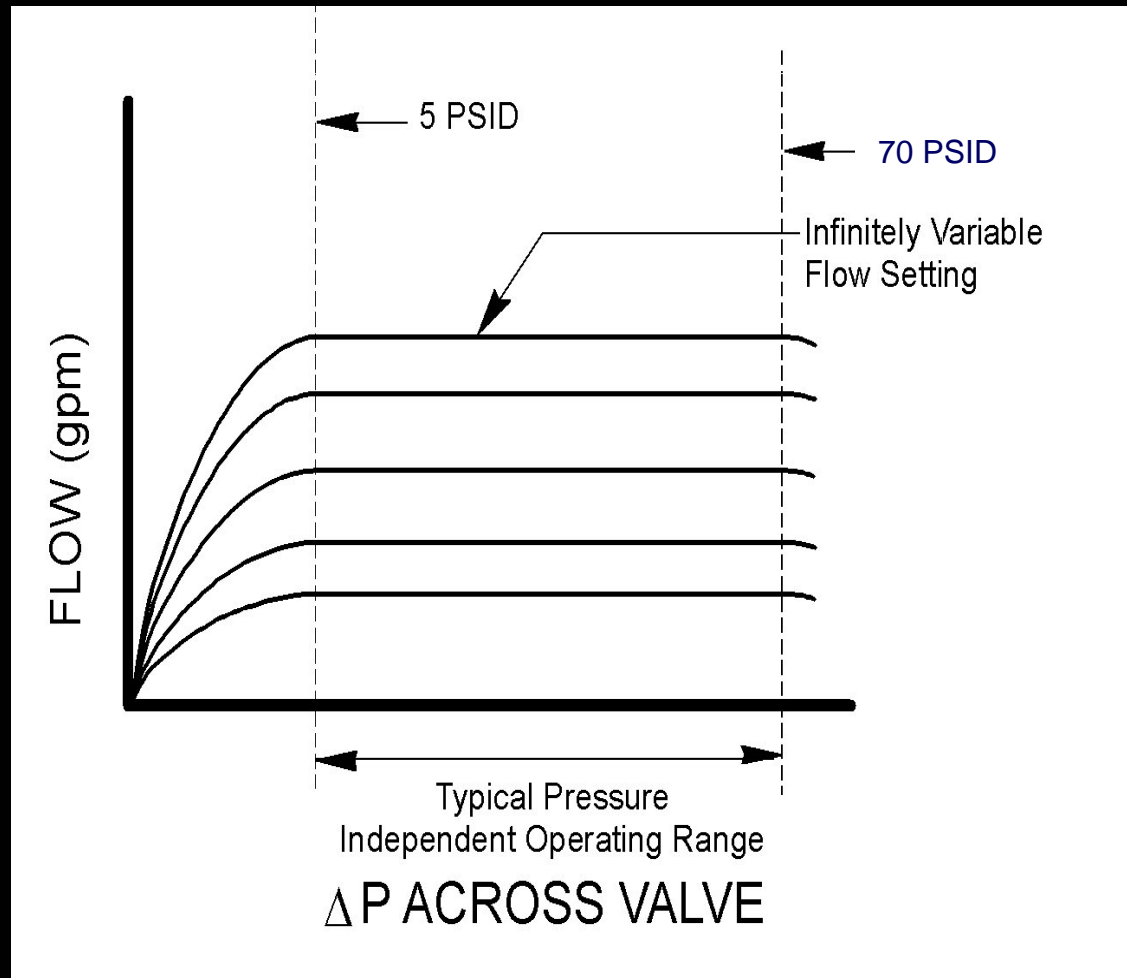
Design Load = 500 GPM

Pressure Independent Control Valve



(note that we no longer need a balance valve)

Flow Rate Doesn't Vary as Differential Pressure Varies



Automatic Flow Limiters –

**Will they solve the
problem?**

Automatic Flow Limiters

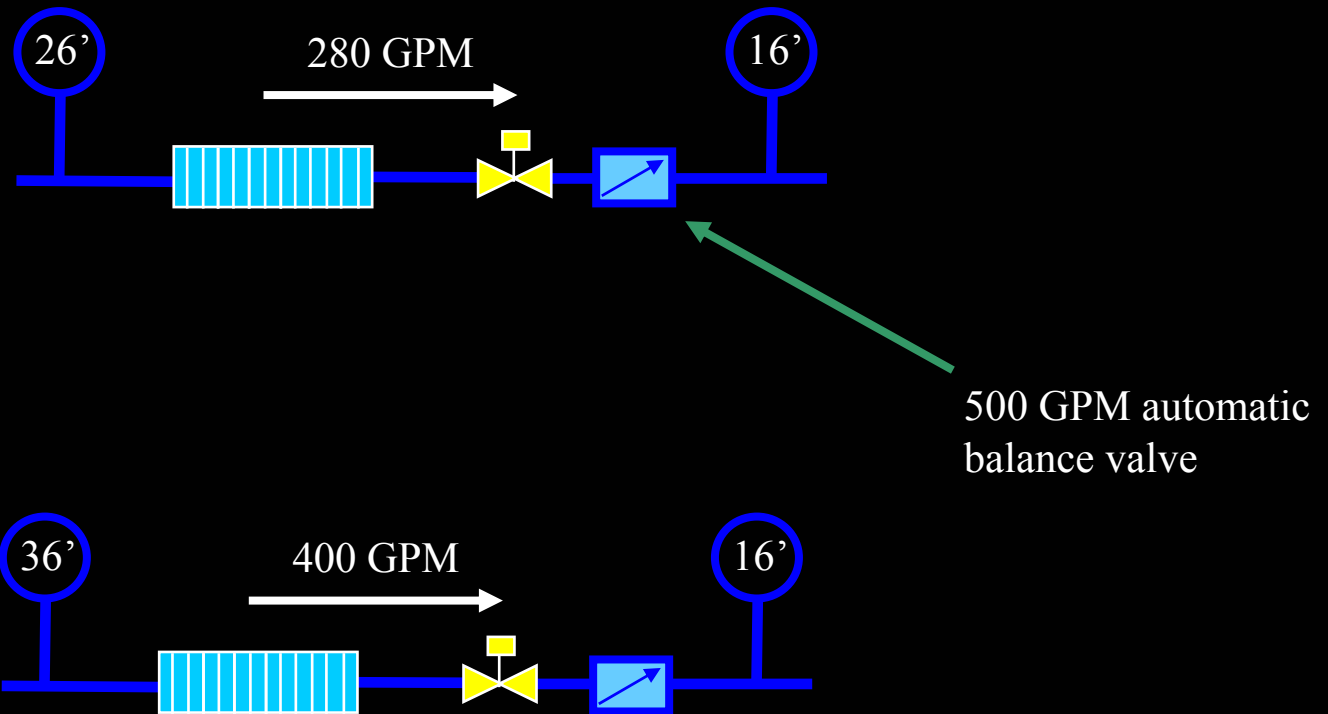
—

**Function as the opposite of
Pressure Independent
Modulating Control Valves!**

Typical Part Load Day

Conventional Control Valve

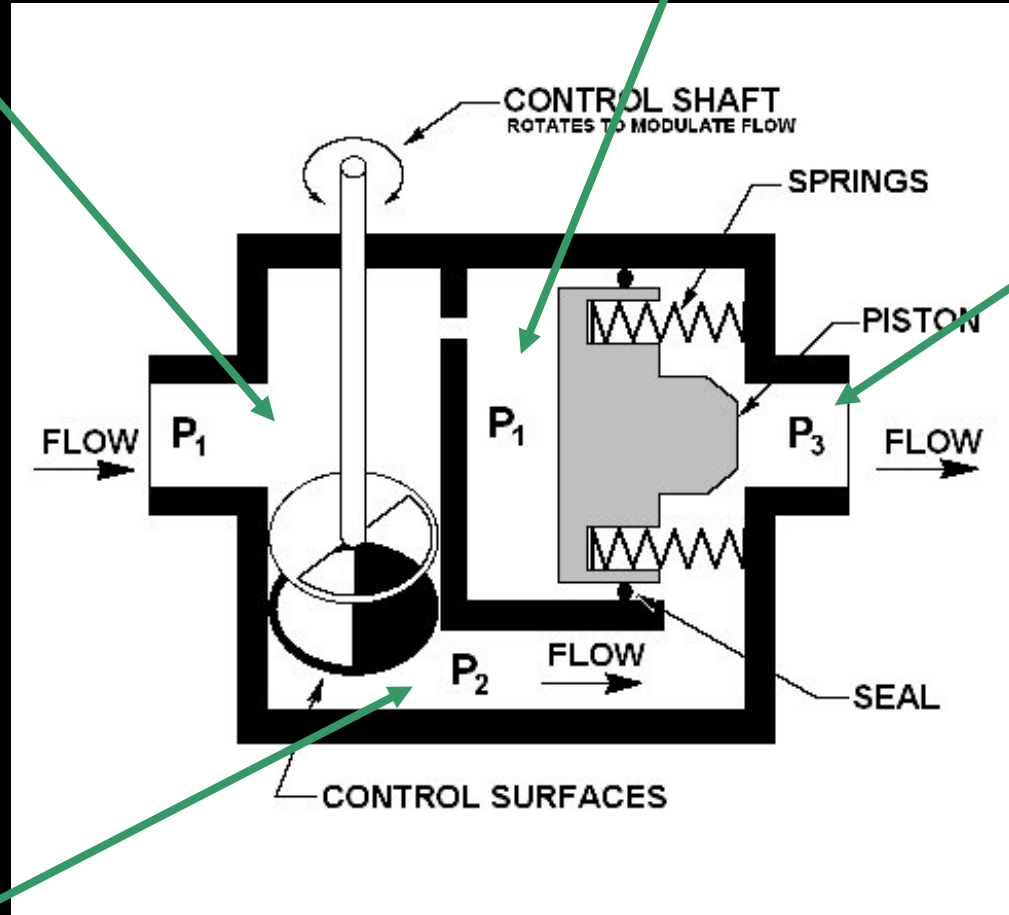
Design Load = 500 GPM



26'

26'

16'

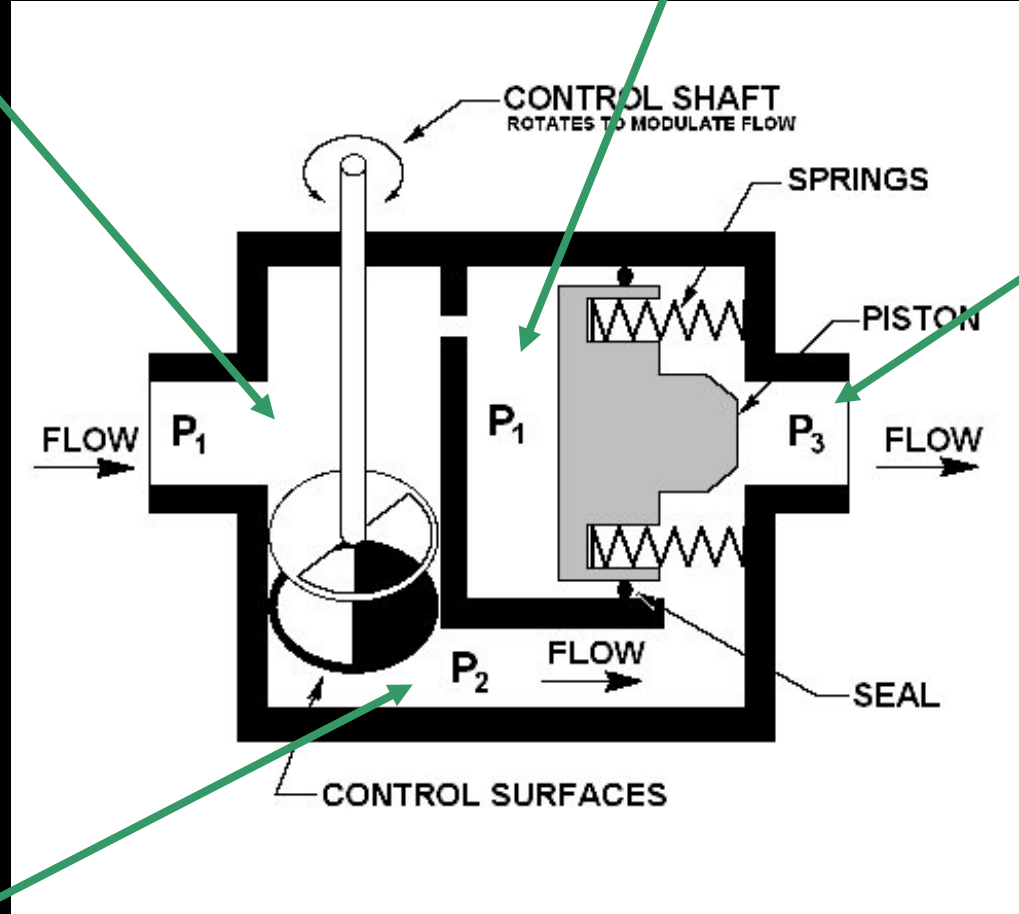


21'

36'

36'

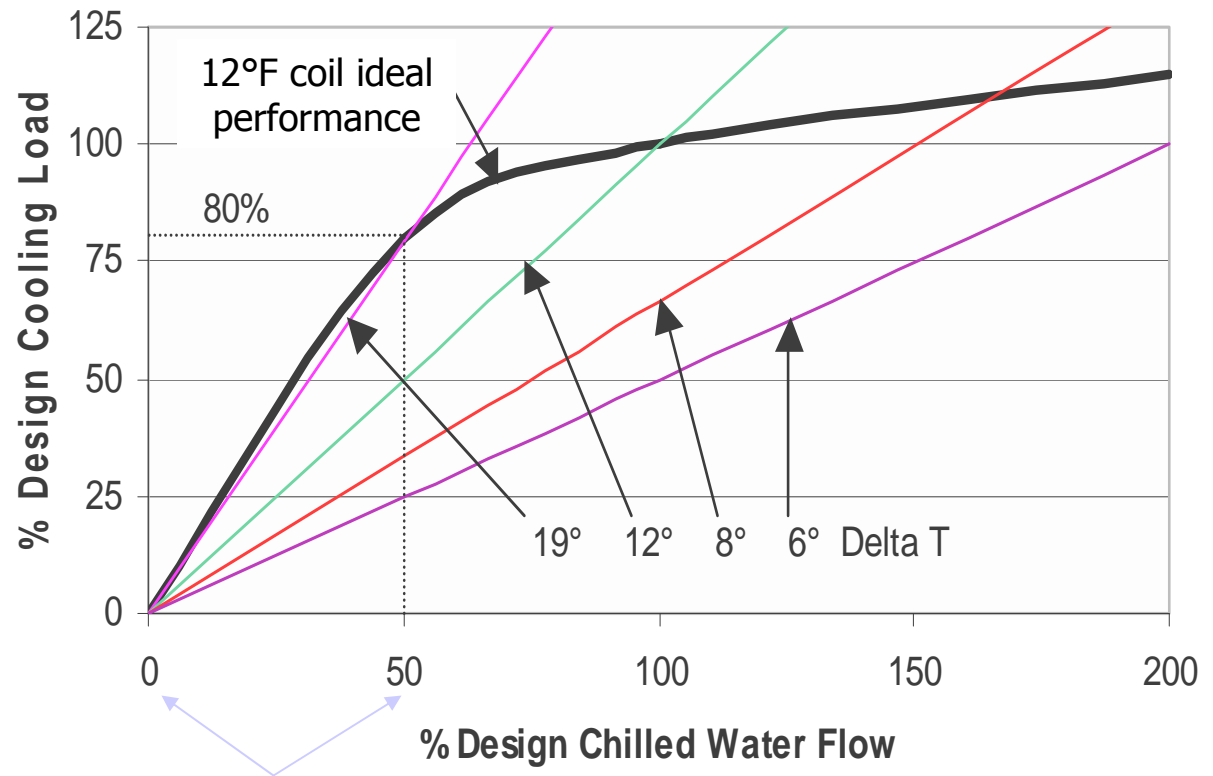
16'



31'

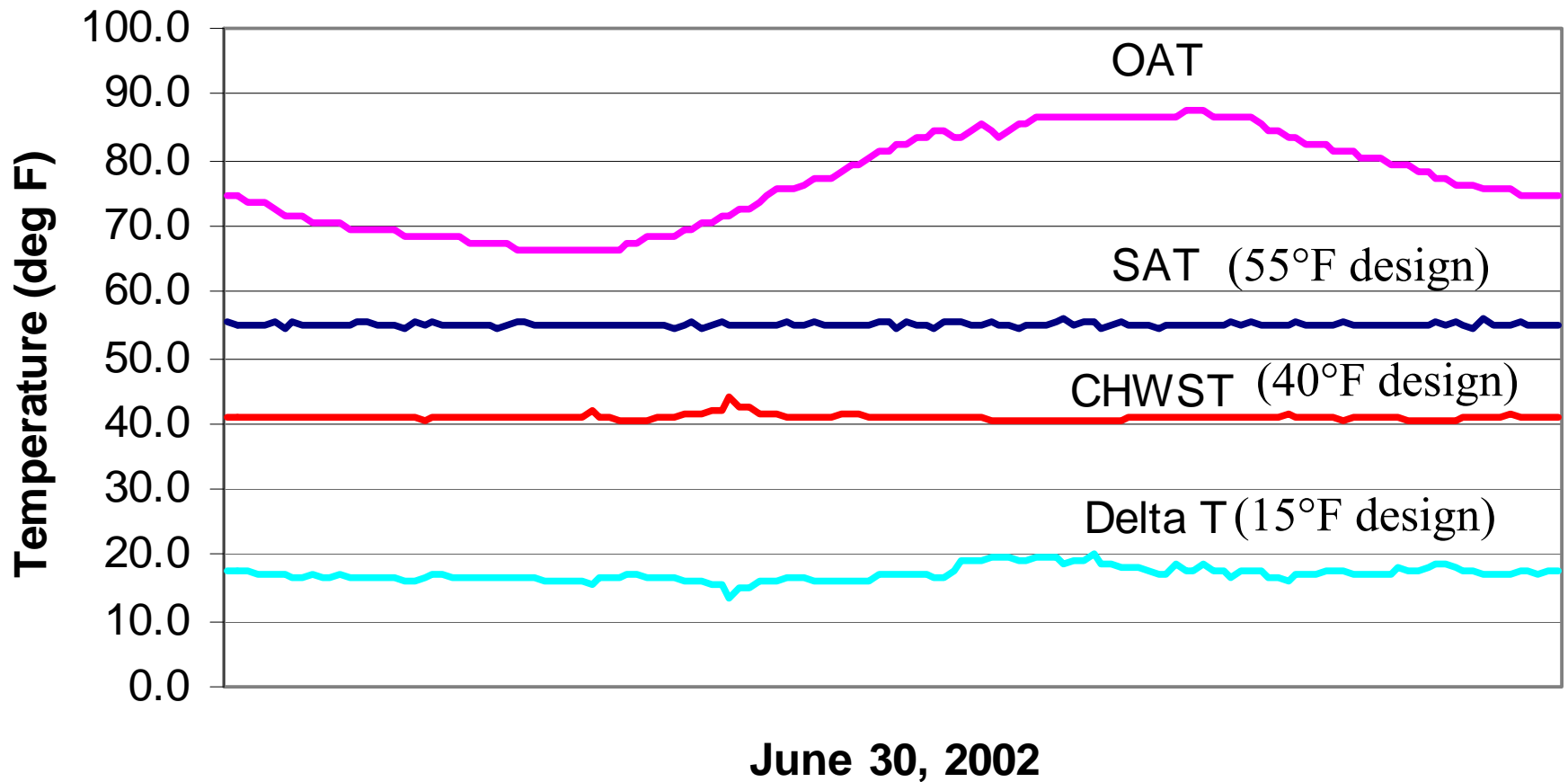
- this enables coil and system ΔT to be optimized at all loads

The BAR is design delta T – the coil delta T should exceed design at all load conditions



Control valves **MUST** be properly sized and able to control in this range the majority of the time.

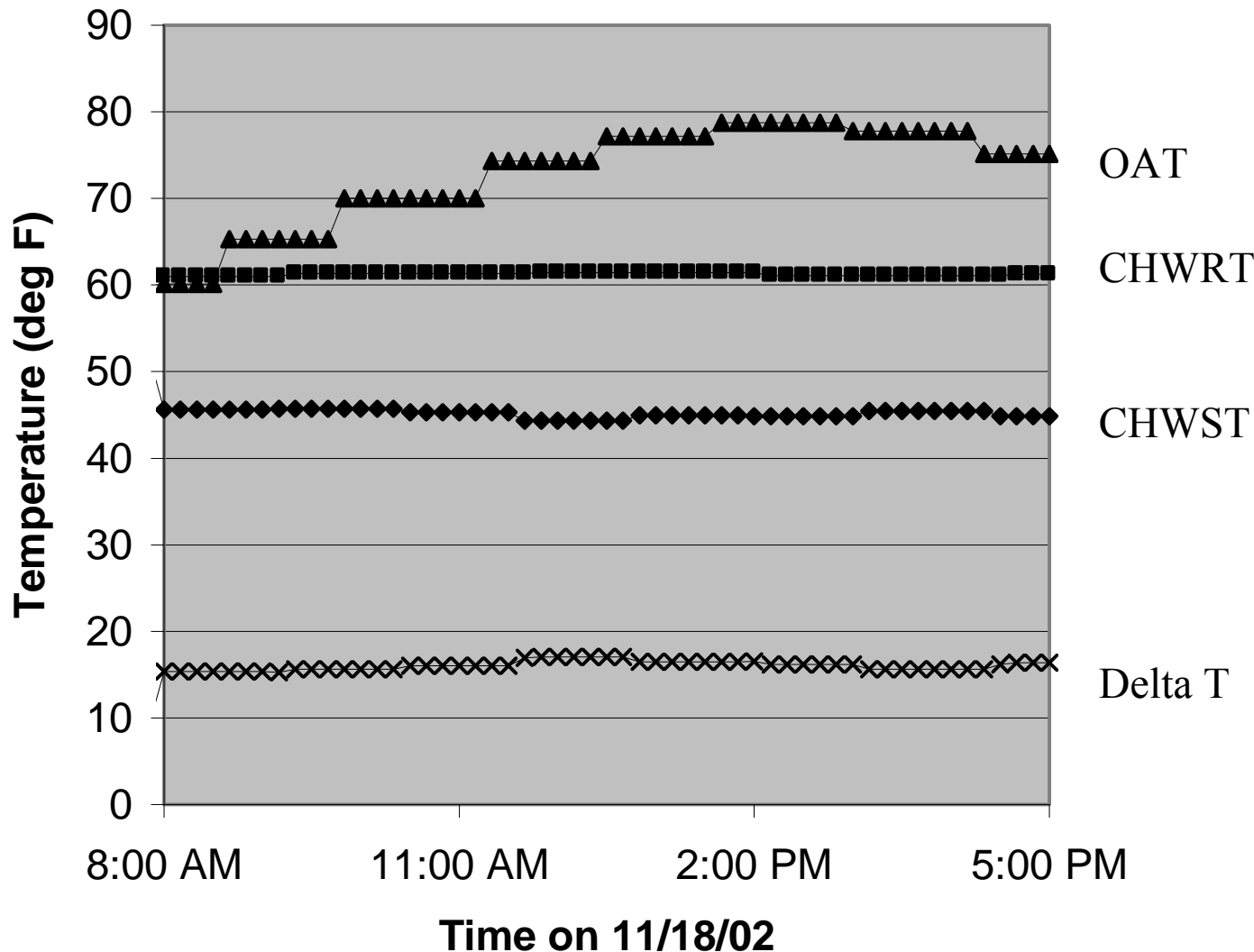
Trend of excellent performance:



Summary – Water Side

- Measure ΔT across coils and all loops.
- Compare ΔT performance to design.
- If it does not exceed coil design at part load
 - seek a solution.

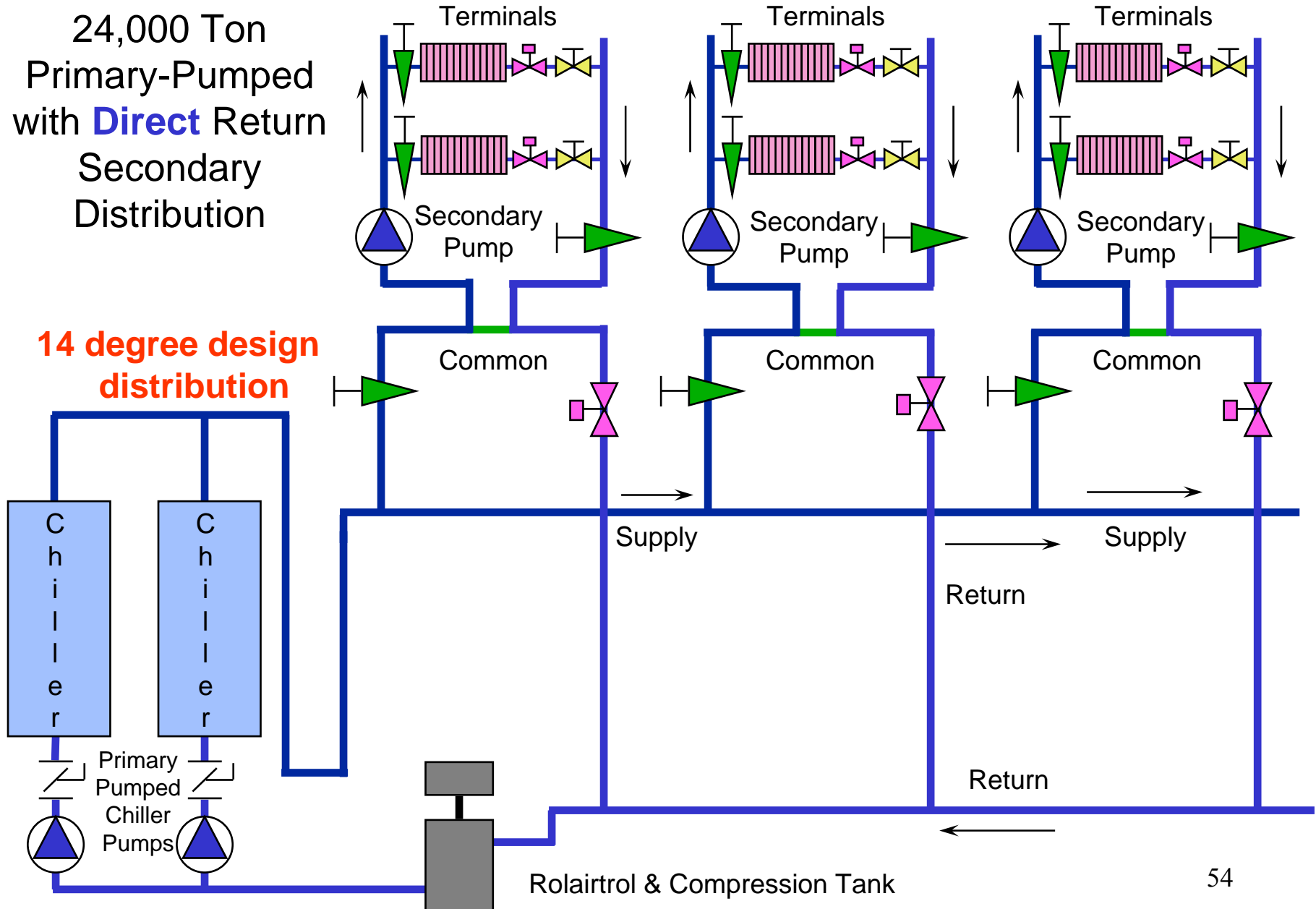
How a good system operates:



USC Law
Building has
10 and 12°F
coils designed
for 44°
CHWST

Duke Chiller Plant
 24,000 Ton
 Primary-Pumped
 with **Direct** Return
 Secondary
 Distribution

Old coils - 10 Degree design



Real World Results at Duke University

**Duke has old two way valves and coils that
were operating at 5 to 6 ° Δ T.**

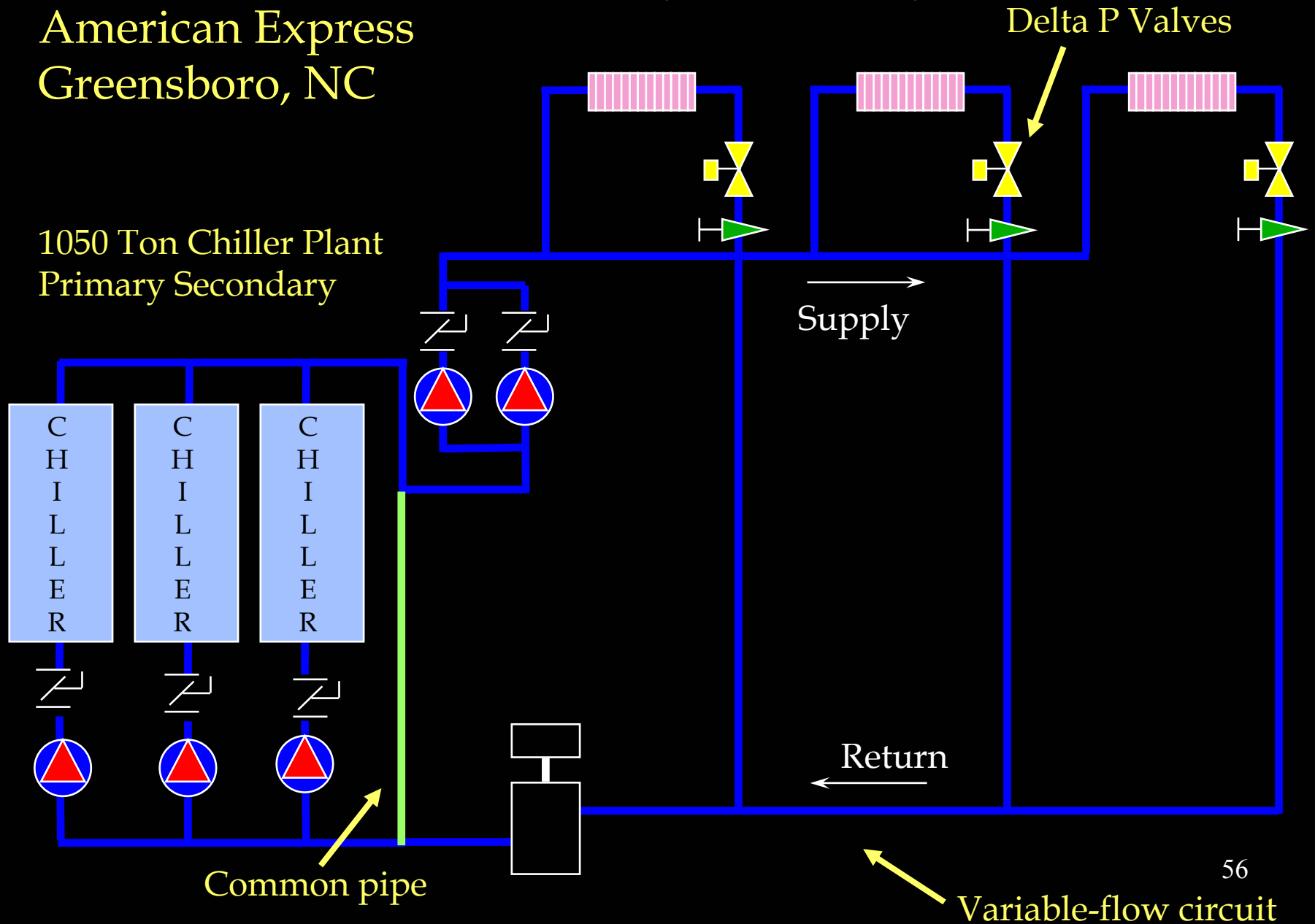
**Pressure Independent Control Valves valves on
same 10 ° coils and loads *are providing*
15 to 20 ° Δ T.**

****If you want to visit call or e-mail the JMP CO***

Primary-Secondary

American Express
Greensboro, NC

1050 Ton Chiller Plant
Primary Secondary

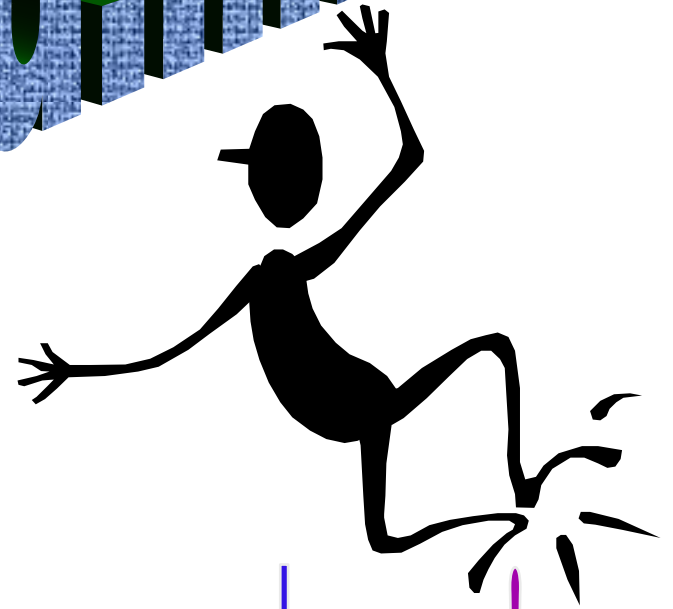


Real World Results at American Express

American Express had old two way valves and coils that were operating at 5 to 6 ° Δ T.

**Only 50% or (1/2 half) valves changed to Pressure Independent Control Valves - on same coils and loads
*are providing 9 to 10 ° Δ T.***

The End



Thank you for coming!

Web Site: <http://www.jmpco.com>